# Introduction to Control (00340040) lecture no. 8

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1/40

#### Outline

Steady-state performance of closed-loop systems and loop shaping

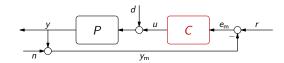
Transient performance of closed-loop systems

Control effort

Loop shaping: what we have and what we miss

Mathematical preliminaries: the (Cauchy's) argument principle

#### Previously on steady-state performance...



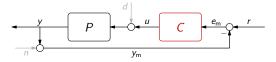
Zero steady-state errors to

- -r(t) = 1(t) requires an integrator in PC
- $-d(t)=\mathbb{I}(t)$  requires an integrator in C
- -r(t) = ramp(t) requires a double integrator in PC
- $-r(t) = \sin(\omega t + \phi)\mathbb{1}(t)$  requires poles at  $\pm j\omega$  in P(s)C(s)
- $-d(t) = \sin(\omega t + \phi)\mathbb{1}(t)$  requires poles at  $\pm j\omega$  in C(s)
- $-n(t) = \sin(\omega t + \phi)\mathbb{1}(t)$  requires zeros at  $\pm j\omega$  in P(s)C(s)

What if  $e_{ss} = 0$  need not be attained? Or if r and / or d have their spectra spread over some frequency range and (uniformly) high-gain feedback is not feasible?

2/40

#### Command following



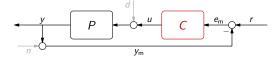
Denote by  $\Omega_r$  the frequency range where the spectrum of r is concentrated. By good steady-state command response we understand that

$$|E(j\omega)| \ll |R(j\omega)|, \quad \forall \omega \in \Omega_r$$

$$\downarrow |S(j\omega)| \ll 1, \quad \forall \omega \in \Omega_r.$$

(remember, e = Sr if d = n = 0).

### Good command following and loop gain



Denote the loop transfer function as L(s) := P(s)C(s), so that

$$S(s) = rac{1}{1 + L(s)} \quad ext{and} \ |S(\mathrm{j}\omega)| \leq rac{1}{|L(\mathrm{j}\omega)| - 1} \ ext{whenever} \ |L(\mathrm{j}\omega)| > 1$$

(by the triangle inequality,  $|L| = |1 + L - 1| \le |1 + L| + 1$ ). Hence,

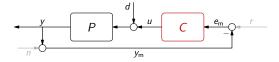
$$|S(j\omega)| \le \sigma_r < 1 \quad \Longleftrightarrow \quad |L(j\omega)| \ge \frac{1+\sigma_r}{\sigma_r} = 1 + \frac{1}{\sigma_r} > 2$$

for every  $\omega \in \Omega_r$ . Qualitatively,

- high loop gain in the whole frequency range  $\omega \in \Omega_r$  guarantees good steady-state command response.

5/40

#### Good disturbance attenuation and loop gain



Now,

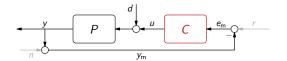
$$|P(\mathrm{j}\omega)| |S(\mathrm{j}\omega)| \le \sigma_d < 1 \quad \Longleftrightarrow \quad |L(\mathrm{j}\omega)| \ge 1 + \frac{|P(\mathrm{j}\omega)|}{\sigma_d}$$

for every  $\omega \in \Omega_d$  (think of bounding S for  $\sigma_r = \sigma_d/|P(j\omega)|$ ). Qualitatively,

- high loop gain in the whole frequency range  $\omega \in \Omega_d$  guarantees good steady-state disturbance attenuation.

Remark: Note that a low plant gain could also help, but this is independent of the choice of the controller *C*.

#### Disturbance attenuation



Denote by  $\Omega_d$  the frequency range where the spectrum of d is concentrated. By good steady-state disturbance attenuation we understand that

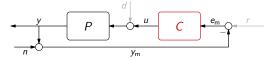
$$|E(j\omega)| = |Y(j\omega)| \ll |D(j\omega)|, \quad \forall \omega \in \Omega_d$$
 $\downarrow \downarrow$ 

$$|T_{\mathsf{d}}(\mathsf{j}\omega)| = |P(\mathsf{j}\omega)| |S(\mathsf{j}\omega)| \ll 1, \quad \forall \omega \in \Omega_d$$

(remember,  $y = -e = T_d d$  if r = n = 0).

6/40

#### Measurement noise sensitivity



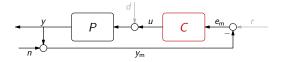
Denote by  $\Omega_n$  the frequency range where the spectrum of n is concentrated. By low steady-state sensitivity to measurement noise we understand that

$$|E(j\omega)| = |Y(j\omega)| \ll |N(j\omega)|, \quad \forall \omega \in \Omega_n$$

$$\downarrow \qquad \qquad |T(j\omega)| \ll 1, \quad \forall \omega \in \Omega_n$$

(remember, y = -e = Tn if r = d = 0).

### Measurement noise sensitivity and high loop gain



Because

$$|T(\mathrm{j}\omega)| = \frac{|L(\mathrm{j}\omega)|}{|1 + L(\mathrm{j}\omega)|} \ge \frac{|L(\mathrm{j}\omega)|}{1 + |L(\mathrm{j}\omega)|},$$

we have that

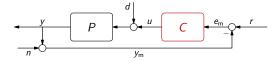
$$|L(j\omega)| > 1 \implies |T(j\omega)| > \frac{1}{2}$$

and as  $|L(j\omega)|$  increases,  $|T(j\omega)| \to 1$ . This means that

- high loop gain does *not* lead to low measurement noise sensitivity.

9/40

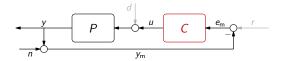
#### Catch-22 situation?



On the one hand,

- we need high loop gain (in  $\omega \in \Omega_r$  and  $\omega \in \Omega_d \setminus \{\omega \mid |P(\mathrm{j}\omega)| \ll 1\}$ ).
- On the other hand,
- we need low loop gain (in ω ∈ Ω<sub>n</sub>).

#### Low measurement noise sensitivity and loop gain



Because T(s) = L(s)/(1 + L(s)) = 1/(1 + 1/L(s)), we have that

$$|T(\mathrm{j}\omega)| \leq rac{1}{1/|L(\mathrm{j}\omega)|-1}$$
 whenever  $|L(\mathrm{j}\omega)| < 1$ .

Hence,

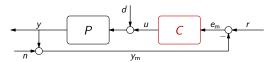
$$|T(j\omega)| \le \sigma_n < 1 \quad \Longleftrightarrow \quad |L(j\omega)| \le \frac{\sigma_n}{1+\sigma_n} \in \left(0, \frac{1}{2}\right)$$

for every  $\omega \in \Omega_n$ . Qualitatively,

- low loop gain in the whole frequency range  $\omega \in \Omega_n$  guarantees low steady-state noise sensitivity.

10/40

### "Typical" spectra of r, d, and n



In many cases<sup>1</sup>,

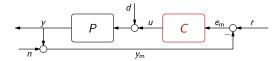
- command signals are "slow" (i.e.  $\Omega_r$  mostly includes low frequencies)
- measurement noise is "fast"
   (i.e. Ω<sub>n</sub> mostly includes high frequencies)

Moreover, since most physical processes are low-pass,

- only "slow" components of d should be worried about ("fast" part of d doesn't show up in y anyway as  $|P(j\omega)| \ll 1$  at high frequencies)

<sup>&</sup>lt;sup>1</sup>Oi va voi if this is not true!

### The first acquaintance with loop shaping



Thus, we may endeavor to design loops with

- high loop gain,  $|L(j\omega)| \gg 1$ , at "low" frequencies
- low loop gain,  $|L(j\omega)| \ll 1$ , at "high" frequencies

where "high" and "low" frequency ranges depend on spectral properties of exogenous signals in the application.

This control design philosophy is called loop shaping.

13/40

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Transient performance of closed-loop systems

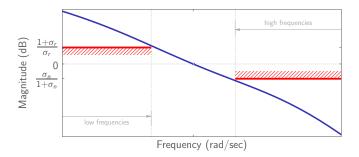
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### Loop shaping: big picture (magnitude)

What we shall try to do is to shape  $|L(j\omega)|$  like this:



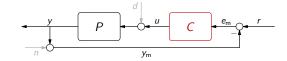
#### Note that

there is always a region where the loop gain is neither high nor low

14/40

#### Closed-loop transient response

We're mostly concerned with transient performance of command response:



and measure it on the basis of the step response (its speed and smoothness).

We know (from Lecture 4) that transient properties in time and frequency domains are related as follows:

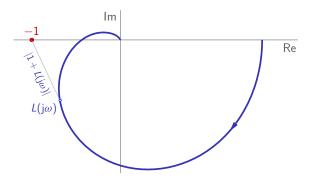
- the wider the bandwidth of  $T(j\omega)$  is, the faster its step response is
- the higher resonant peaks of  $T(j\omega)$  are, the larger over / undershoot is

#### The question:

- could these requirements be expressed in terms of  $L(j\omega)$ ?

### Closed vs. open loop: resonant peak of T

Given  $\omega$ ,  $|1 + L(j\omega)|$  is the distance between the points  $L(j\omega)$  and -1 + j0 in the complex plane of  $L(j\omega)$ :

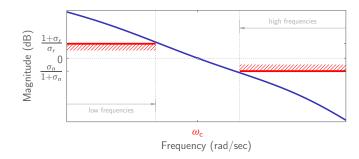


Thus,

- the closer  $L(j\omega)$  to -1+j0 is, the larger  $|T(j\omega)|=\frac{|L(j\omega)|}{|1+L(j\omega)|}$  is (as  $L(j\omega)$  approaches -1+j0, magnitude  $|L(j\omega)|\to 1$ ).

17/40

### Crossover frequency and crossover region



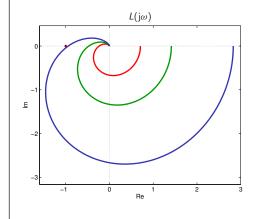
When  $\omega$  increases,  $L(j\omega)$  passes from the low- to high-frequency range. On its way it necessarily passes the area with  $|L(j\omega)|\approx 1$ . This frequency range is called the crossover region and the frequency  $\omega$  at which  $|L(j\omega)|=1$  is called the crossover frequency and denoted  $\omega_{\rm c}$ , i.e.

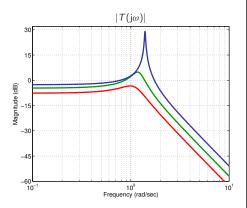
$$|L(j\omega_{c})|=1.$$

There may be more than one crossover frequencies.

### Closed vs. open loop: example

Let 
$$L(s) = \frac{k\sqrt{2}}{(s+1)(s^2+s+1)}$$
. Then for  $k \in \{0.5, 1, 2\}$  we have:



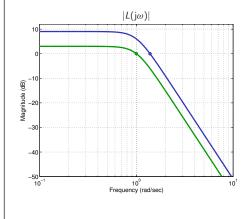


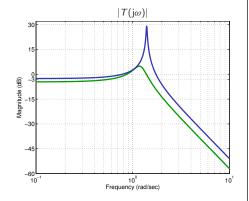
18/4

### Closed vs. open loop: bandwidth of T

The closed-loop bandwidth  $\omega_b$  is typically close to the crossover frequency  $\omega_c$ . A rule of thumb is that  $\omega_b \approx 1.2 \div 1.5\omega_c$ .

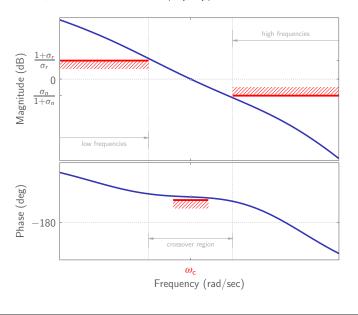
For example, for  $L(s)=\frac{k\sqrt{2}}{(s+1)(s^2+s+1)}$  and  $k\in\{1,2\}$  we have:





### Loop shaping: big picture (more details will follow)

What we shall try to do is to shape  $|L(j\omega)|$  like this:



21/40

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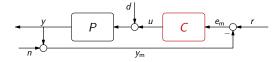
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22/40

### Closed-loop control signal



Remember, from Lecture 5, that

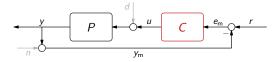
$$u = T_c r - Td - T_c n$$
.

Thus, properties of the control signal in closed-loop control are

- shaped by properties of  $T_{\rm c}$  and T

(rather than by properties of  $C_{ol}$  as in the open-loop case).

### Steady-state control effort: command response



Since

$$|T_{c}(j\omega)| = \frac{|T(j\omega)|}{|P(j\omega)|},$$

properties of  $\boldsymbol{u}$  are again

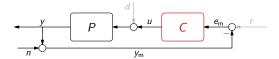
 $-\,$  determined by relations between controlled & uncontrolled bandwidths.

For example, if both P and T are low-pass filters and  $\omega_{b,T} > \omega_{b,P}$ , we may expect that  $|T_c(j\omega)|$  grows in  $\omega \in (\omega_{b,P}, \omega_{b,T})$ . Hence,

-  $\omega_{b,T}\gg\omega_{b,P}$  would lead to  $|\mathcal{T}_{c}(j\omega)|\gg1$  around  $\omega_{b,T}$ , which might not be affordable.

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#### Steady-state control effort: measurement noise response

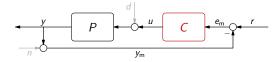


The growth of  $|T_c(j\omega)|$  at high frequencies is

- even more dangerous from the noise response perspective as this might be where the spectrum of the noise n is concentrated. Having high-magnitude high-frequency oscillations of u is highly undesirable as it
- might harm controlled process / actuators
- might excite poorly modeled high-frequency modes of the plant

25/40

### Control effort during transients

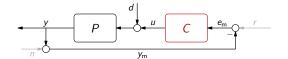


An additional side effect of reaching  $\omega_{b,T} \gg \omega_{b,P}$ 

 $-T_c(i\omega)$  has high-frequency resonant peak(s),

which, in turn, leads to high-amplitude peaks in the step response of u (like in open-loop control, cf. discussion in Lecture 5).

#### Steady-state control effort: disturbance response



As we anyway should

- aim at avoiding high magnitude of  $T(j\omega)$ , the effect of the input disturbances on the control signal
- needs no special attention.

26/4

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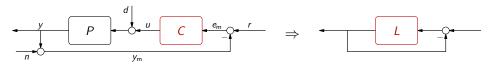
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#### What did we learn by now

We may replace



#### and aim at

- having appropriate crossover frequency,  $\omega_c$ 
  - high enough: to cover spectra of reference / disturbances and have sufficiently fast transients
  - not too high: to avoid the amplification of measurement noise and excessive grows of the control signal
- high loop gain,  $|L(j\omega)|\gg 1$ , at low frequencies  $(\omega\ll\omega_{
  m c})$
- low loop gain,  $|L(j\omega)| \ll 1$ , at high frequencies  $(\omega \gg \omega_c)$
- keeping  $L(j\omega)$  "far" from the point -1 + j0 in the crossover region

29/40

#### Stability analysis



We can analyze the stability of this system by

- algebraic analysis of the closed-loop characteristic polynomial
- graphical root-locus analysis

#### But

 $\ddot{}$  neither of them does it in terms of the frequency response of L, which is what loop shaping needs.

#### Don't we miss something important?



- \_ ...
- having appropriate crossover frequency,  $\omega_{\mathsf{c}}$
- high loop gain,  $|L(j\omega)| \gg 1$ , at low frequencies  $(\omega \ll \omega_c)$
- low loop gain,  $|L(j\omega)| \ll 1$ , at high frequencies  $(\omega \gg \omega_c)$
- keeping  $L(j\omega)$  "far" from the critical point in the crossover region

Of course.

## stability

30/4

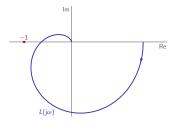
#### Nyquist stability criterion: what does it offer



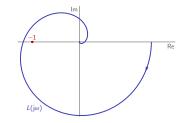
It analyzes the stability of the closed-loop system on the basis of

- number of unstable pole of L(s) and
- position of the polar plot of  $L(j\omega)$  with respect to the point -1 + j0

For example, if L is stable, then



closed-loop system is stable



closed-loop system is unstable

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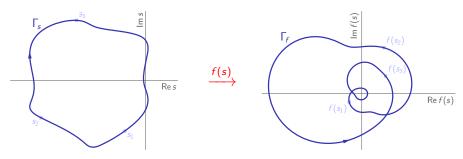
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33/40

#### Mapping contours in s-plane

Similarly, an s-plane contour  $\Gamma_s$  lying in the domain of f(s) is mapped to a contour  $\Gamma_f$  in the f(s)-plane:



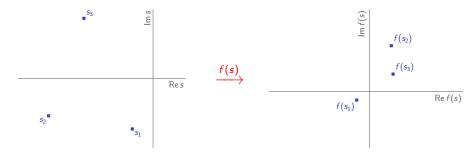
A contour is said to be

- simple if it does not intersect itself and
- closed if it starts and ends at the same point

( $\Gamma_s$  above is simple closed, whereas  $\Gamma_f$  is closed yet not simple).

#### Mapping points in s-plane

Consider a complex function f(s). For any  $s \in \mathbb{C}$  from its domain,  $f(s) \in \mathbb{C}$  too. We say that s is mapped by f from the s-plane to the f(s)-plane:



(here 
$$f(s) = \frac{0.273(-s+0.2)}{(s+0.6)(s+1)(s+1.3)}$$
, if you're curious).

34/40

#### The argument principle

Let

- $-\Gamma_s$  be a simple closed contour,
- -f(s) be meromorphic (i.e. only poles as singularities) inside and on  $\Gamma_s$ ,
- $Z_f$  be the number of zeros of f(s) inside  $\Gamma_s$ ,
- $P_f$  be the number of poles of f(s) inside  $\Gamma_s$ .

#### Theorem (Cauchy)

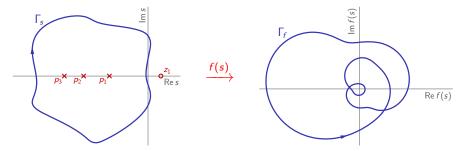
If  $\Gamma_s$  passes through neither poles nor zeros of f(s), then  $\Gamma_f$  encircles the origin  $Z_f - P_f$  times in the clockwise direction as s traverses  $\Gamma_s$  in the clockwise direction.

#### The argument principle: example 1

Consider

$$f(s) = \frac{0.273(-s+0.2)}{(s+0.6)(s+1)(s+1.3)},$$

which has  $P_f = 3$  poles and  $Z_f = 0$  zeros inside  $\Gamma_s$ .



Hence,  $\Gamma_f$  encircles the origin -3 times<sup>2</sup> in the clockwise direction.

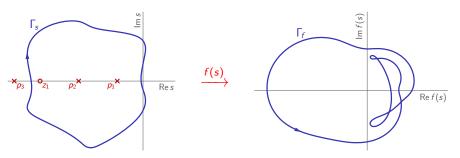
37 /40

### The argument principle: example 3

Consider

$$f(s) = -\frac{0.7(s+1.6)}{(s+0.4)(s+1)(s+2)},$$

which has  $P_f = 2$  poles and  $Z_f = 1$  zero inside  $\Gamma_s$ .



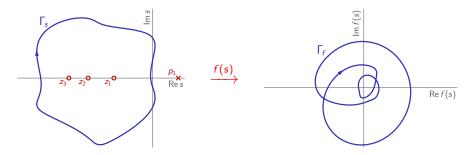
Hence,  $\Gamma_f$  encircles the origin -1 time<sup>3</sup> in the clockwise direction.

The argument principle: example 2

Consider

$$f(s) = \frac{0.13(s+0.6)(s+1)(s+1.3)}{-s+0.4},$$

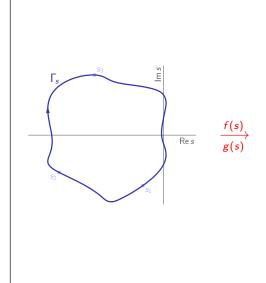
which has  $P_f = 0$  poles and  $Z_f = 3$  zeros inside  $\Gamma_s$ .

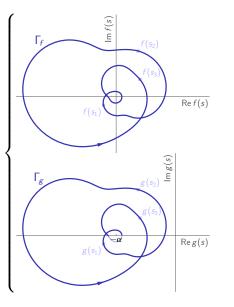


Hence,  $\Gamma_f$  encircles the origin 3 times in the clockwise direction.

#### Shift by a constant

Let  $f(s) = \alpha + g(s)$  for some  $\alpha \in \mathbb{R}$ . Then





<sup>&</sup>lt;sup>2</sup>That is, 3 times in the counterclockwise direction.

<sup>&</sup>lt;sup>3</sup>That is, once in the counterclockwise direction.