## Linear Systems (034032)

lecture no. 7

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## LTI systems response: from time to Laplace / z domain

We know that if $G: u \mapsto y$ is LTI (linear time invariant), then

$$
y=g * u
$$

where $g=G \delta$ is the impulse response of $G$, i.e. its response to $\delta$ applied at $t=0$. By the convolution property of the Laplace $/ z$ transform,

$$
Y(s)=G(s) U(s) \quad \text { or } \quad Y(z)=G(z) U(z)
$$

In other words, (dynamic) LTI systems in the Laplace / z domain act as the (static) multiplication of the transformed impulse response and input. The function

$$
G(s)=(\mathfrak{L}\{g\})(s) \quad \text { or } \quad G(z)=(\mathfrak{Z}\{g\})(z)
$$

is called the transfer function of $G$. Transfer function may also be viewed as - the ratio of the Laplace / $z$ transforms of the output and input signals,

$$
G(s)=\frac{Y(s)}{U(s)} \quad \text { or } \quad G(z)=\frac{Y(z)}{U(z)}
$$

## Outline

## Transfer functions of LTI systems

## Transfer functions: examples

Gain: its impulse response $g=k \delta$, so $G(s)=k$ and $G(z)=k$.

Delay: its impulse response $g=\mathbb{S}_{-\tau} \delta$, so $G(s)=\mathrm{e}^{-\tau s}$ and $G(z)=z^{-\tau}$.
Integrator: its impulse response $g_{\text {int }}=\mathbb{1}$, so

$$
G_{\mathrm{int}}(s)=\frac{1}{s} \quad \text { and } \quad G_{\mathrm{int}}(z)=\frac{z}{z-1}
$$

has a single pole at $s=0$ and $z=1$, respectively.
Finite-memory integrator: its impulse response $g_{f m i, \mu}=\$_{-\mu / 2}$ rect $_{\mu}$, so

$$
G_{\mathrm{fmi}, \mu}(s)=\frac{1-\mathrm{e}^{-\mu s}}{s}=\int_{0}^{\mu} \mathrm{e}^{-s t} \mathrm{~d} t
$$

is an entire function.

## Transfer function description

The fact that dynamic LTI systems can be described by algebraic relations,

$$
Y(s)=G(s) U(s) \quad \text { or } \quad Y(z)=G(z) U(z)
$$

is a handy property, having a potential of greatly simplifying the analysis. It has no counterparts in the realm of general time-varying/nonlinear systems, making the class of LTI systems so special and sought after.

## Still, we yet to know how to

- express properties of LTI systems in terms of their transfer functions, without which algebraic relations are not more than a curiosity.


## System interconnections via transfer functions

Parallel:


Cascade: If $G=\wedge G_{2} \cdot G_{1} \cdot u$, then $G(s)=G_{2}(s) G_{1}(s) \quad$ product

Feedback: If $G=$
by

$$
Y=G_{1}\left(U \pm G_{2} Y\right) \Longleftrightarrow\left(1 \mp G_{1} G_{2}\right) Y=G_{1} U
$$

## $L_{2} / \ell_{2}$ stability via transfer functions

## Theorem

A continuous-time LTI system is causal and $L_{2}$-stable iff its transfer function is holomorphic and bounded in $\mathbb{C}_{0}:=\{s \in \mathbb{C} \mid \operatorname{Re} s>0\}$.

Theorem
A discrete-time $L T I$ system is causal and $\ell_{2}$-stable iff its transfer function is holomorphic and bounded in $\{z \in \mathbb{C}||z|>1\}$.

Example 1: we know that $G_{\mathrm{fmi}, \mu}(s)=\int_{0}^{\mu} \mathrm{e}^{-s t} \mathrm{~d} t$, which is entire. Now,

$$
\left|G_{\mathrm{fmi}, \mu}(s)\right|=\left|\int_{0}^{\mu} \mathrm{e}^{-s t} \mathrm{~d} t\right| \leq \int_{0}^{\mu}\left|\mathrm{e}^{-s t}\right| \mathrm{d} t=\int_{0}^{\mu} \mathrm{e}^{-\operatorname{Re} s t} \mathrm{~d} t<\int_{0}^{\mu} \mathrm{d} t=\mu
$$

for all $\operatorname{Re} s>0$. Thus, $G_{f m i, \mu}(s)$ is not only holomorphic, but also bounded in $\mathbb{C}_{0}$, meaning that $G_{f m i, \mu}$ is $L_{2}$-stable.
Example 2: Consider $G: u \mapsto y$ acting as $y(t)=\dot{u}(t)$. Its transfer function $G(s)=s$ is entire, but not bounded in $\mathbb{C}_{0}$, meaning that $G$ is not $L_{2}$-stable.

## Outline

LTI systems with rational transfer functions

## LTI systems described by ODE

Let $G: u \mapsto y$ be an analog LTI system described as

$$
\begin{aligned}
& y^{(n)}(t)+a_{n-1} y^{(n-1)}(t)+\cdots+a_{1} \dot{y}(t)+a_{0} y(t) \\
& \quad=b_{m} u^{(m)}(t)+b_{m-1} u^{(m-1)}(t)+\cdots+b_{1} \dot{u}(t)+b_{0} u(t)
\end{aligned}
$$

for some $a_{i}, b_{i} \in \mathbb{R}$. In the Laplace domain this relation reads

$$
Y(s)=\frac{b_{m} s^{m}+b_{m-1} s^{m-1}+\cdots+b_{1} s+b_{0}}{s^{n}+a_{n-1} s^{n-1}+\cdots+a_{1} s+a_{0}} U(s)
$$

Hence, the transfer function of this system is

$$
G(s)=\frac{b_{m} s^{m}+b_{m-1} s^{m-1}+\cdots+b_{1} s+b_{0}}{s^{n}+a_{n-1} s^{n-1}+\cdots+a_{1} s+a_{0}} .
$$

In other words,

- transfer functions of LTI systems described by ODE are always rational (actually, real-rational if the coefficients are real).


## Interconnections

If

$$
G_{1}(s)=\frac{N_{1}(s)}{D_{1}(s)} \quad \text { and } \quad G_{2}(s)=\frac{N_{2}(s)}{D_{2}(s)},
$$

then

$$
\begin{aligned}
G_{1}(s)+G_{2}(s) & =\frac{N_{1}(s)}{D_{1}(s)}+\frac{N_{2}(s)}{D_{2}(s)}=\frac{N_{1}(s) D_{2}(s)+N_{2}(s) D_{1}(s)}{D_{1}(s) D_{2}(s)} \\
G_{2}(s) G_{1}(s) & =\frac{N_{2}(s)}{D_{2}(s)} \frac{N_{1}(s)}{D_{1}(s)}=\frac{N_{2}(s) N_{1}(s)}{D_{2}(s) D_{1}(s)} \\
\frac{G_{1}(s)}{1 \mp G_{1}(s) G_{2}(s)} & =\frac{\frac{N_{1}(s)}{D_{1}(s)}}{1 \mp \frac{N_{1}(s)}{D_{1}(s)} \frac{N_{2}(s)}{D_{2}(s)}}=\frac{\frac{N_{1}(s)}{D_{1}(s)}}{\frac{D_{1}(s) D_{2}(s) \mp N_{1}(s) N_{2}(s)}{D_{1}(s) D_{2}(s)}} \\
& =\frac{N_{1}(s) D_{2}(s)}{D_{1}(s) D_{2}(s) \mp N_{1}(s) N_{2}(s)}
\end{aligned}
$$

are rational as well. In other words,

- parallel, cascade, \& feedback interconnections of systems with rational transfer functions result in systems with rational transfer functions.


## Impulse response of continuous-time systems

Let $G$ be an LTI system with a rational proper transfer function of the form

$$
G(s)=\frac{b_{m} s^{m}+b_{m-1} s^{m-1}+\cdots+b_{1} s+b_{0}}{s^{n}+a_{n-1} s^{n-1}+\cdots+a_{1} s+a_{0}}, \quad n \geq m
$$

Let it have $k$ distinct poles $p_{i} \in \mathbb{C}$ of order $n_{i} \in \mathbb{N}$. In this case

$$
G(s)=G(\infty)+\sum_{i=1}^{k} \sum_{j=1}^{n_{i}} \frac{c_{i j}}{\left(s-p_{i}\right)^{j}}
$$

The inverse Laplace transforms $g_{i j}$ of $G_{i j}(s):=c_{i j} /\left(s-p_{i}\right)^{j}$ are

$$
g_{i 1}(t)=c_{i 1} \mathrm{e}^{p_{i} t} \mathbb{T}(t) \quad \text { and } \quad g_{i j}(t)=\frac{c_{i j}}{(j-1)!} t^{j-1} \mathrm{e}^{p_{i} t} \mathbb{1}(t), \quad j>1
$$

(by the $t$-modulation property). Therefore, by linearity

$$
g(t)=G(\infty) \delta(t)+\sum_{i=1}^{k}\left(c_{i 1}+\sum_{j=2}^{n_{i}} \frac{c_{i j}}{(j-1)!} t^{j-1}\right) \mathrm{e}^{p_{i} t} \mathbb{1}(t)
$$

i.e. it is a superposition of ( $t$-modulated) exponential functions.

## Impulse response of continuous-time systems: example

Consider

$$
\begin{aligned}
G(s) & =\frac{s^{2}-2 s+6}{s^{4}+5 s^{3}+8 s^{2}+6 s}=\frac{s^{2}-2 s+6}{s(s+3)\left(s^{2}+2 s+2\right)} \\
& =\frac{\operatorname{Res}(G, 0)}{s}+\frac{\operatorname{Res}(G,-3)}{s+3}+\frac{\operatorname{Res}(G,-1+\mathrm{j})}{s+1-\mathrm{j}}+\frac{\operatorname{Res}(G,-1-\mathrm{j})}{s+1+\mathrm{j}} \\
& =\frac{1}{s}-\frac{1.4}{s+3}+\frac{0.2+1.4 \mathrm{j}}{s+1-\mathrm{j}}+\frac{0.2-1.4 \mathrm{j}}{s+1+\mathrm{j}} \\
& =\frac{1}{s}-\frac{1.4}{s+3}+\frac{0.4 s-2.8}{s^{2}+2 s+2}=\frac{1}{s}-\frac{1.4}{s+3}+2 \sqrt{2} \frac{\frac{\sqrt{2}}{10}(s+1)-\frac{7 \sqrt{2}}{10}}{(s+1)^{2}+1}
\end{aligned}
$$

Hence, taking into account that $\arccos (-0.7 \sqrt{2})=2.999695599 \approx 3$,

$$
g(t)=\mathbb{1}(t)-1.4 \mathrm{e}^{-3 t} \mathbb{1}(t)+2 \sqrt{2} \mathrm{e}^{-t} \sin (t+3) \mathbb{1}(t)
$$



## Impulse response of continuous-time systems (contd)

If
$-\operatorname{Re} p_{i} \geq 0$, then $\sum_{j=1}^{n_{i}} g_{i j} \notin L_{1}$
(terms do not decay)
$-\operatorname{Re} p_{i}<0$, then

$$
\int_{0}^{\infty}\left|\frac{c_{i j}}{(j-1)!} t^{j-1} \mathrm{e}^{p_{i} t}\right| \mathrm{d} t=\frac{\left|c_{i j}\right|}{\left(-\operatorname{Re} p_{i}\right)^{j}} \quad \Longrightarrow \quad \sum_{j=1}^{n_{i}} g_{i j} \in L_{1}
$$

If $\operatorname{Im} p_{i_{1}} \neq 0$, then $\exists i_{2} \neq i_{1}$ such that $p_{i_{2}}=\overline{p_{i_{1}}}$ and also $c_{i_{2} 1}=\overline{c_{i_{1} 1}}$. Then

$$
\begin{aligned}
\frac{c_{i_{1} 1}}{s-p_{i_{1}}}+\frac{c_{i_{2} 1}}{s-p_{i_{2}}} & =\frac{c_{i_{1} 1}}{s-p_{i_{1}}}+\frac{\overline{c_{i_{1} 1}}}{s-\overline{p_{i_{1}}}}=\frac{c_{i_{1} 1}\left(s-\overline{p_{i_{1}}}\right)+\overline{c_{i_{1} 1}}\left(s-p_{i_{1}}\right)}{\left(s-p_{i_{1}}\right)\left(s-\overline{p_{i_{1}}}\right)} \\
& =\frac{2 \operatorname{Rec} c_{i_{1} 1}\left(s-\operatorname{Re} p_{i_{1}}\right)-2 \operatorname{Im} c_{i_{1} 1} \operatorname{Im} p_{i_{1}}}{\left(s-\operatorname{Re} p_{i_{1}}\right)^{2}+\left(\operatorname{Im} p_{i_{1}}\right)^{2}}
\end{aligned}
$$

Thus, if all poles are simple, then we can always end up with an alternative partial fraction expansion with 1 - and 2-order real-rational terms only. And we know that they correspond to exponentials and (modulated) sine waves.

## Impulse response of discrete-time systems

Rational transfer functions in the $z$-domain are also decomposed to partial fractions. Time responses are then also superposition of exponential terms. However, discrete-time responses of corresponding terms are

- way messier than their continuous-time counterparts and less intuitive, with some additional properties, like possibly oscillatory response of 1-order terms. For those reasons we won't discuss details.


## I/O stability via transfer functions

## Theorem

If the transfer function $G(s)$ of a continuous-time LTI system $G$ is rational, then $G$ is causal and I/O stable iff

- $G(s)$ is proper and has no poles in $\overline{\mathbb{C}}_{0}:=\{s \in \mathbb{C} \mid \operatorname{Re} s \geq 0\}$


## Theorem

If the transfer function $G(z)$ of a discrete-time $L T I$ system $G$ is rational, then

- $G$ is causal iff $G(z)$ is proper and
- $G$ is $I / O$ stable iff $G(z)$ has no poles in $\mathbb{C} \backslash \mathbb{D}_{1}=\{z \in \mathbb{C}| | z \mid \geq 1\}$

Example 2 (contd): The differentiator $y(t)=\dot{u}(t)$ has the rational transfer function $G(s)=s$. But it is not proper, attesting to what we already saw.
Example 3: The discrete system $G: u \mapsto y$ acting as $y[t]=u[t+4]-u[t]$ has the (rational) transfer function $G(z)=z^{4}-1$. This $G(z)$ is non-proper and has no poles. Hence, $G$ is not causal, but is stable.

## Persistent test signals

Responses of systems obviously vary depending on inputs. It is not realistic to have a universal understanding of responses of dynamic systems to every possible input. Instead, we shall try to understand responses to (relatively) simple signals, which may nevertheless be representative in many situations (unlike the impulse). Such signals, termed test signals, normally

- have support in $\mathbb{R}_{+}$(so the action starts at $t=0$ )
- are persistent (i.e. do not decay)
- have some regularity (like convergent or periodic)

Some commonly used test signals:
$-\operatorname{step} u(t)=\mathbb{1}(t)$
represents a quick change of an otherwise constant or slowly varying environment

- sine wave $u(t)=\sin \left(\omega_{u} t+\phi_{u}\right) \mathbb{1}(t)$
- T-periodic
with only pure imaginary poles as singularities of their Laplace transforms.


## Outline

Steady-state and transient responses

## Steady-state and transients of the step responses

Consider a continuous-time LTI $G$ with a rational transfer function. If $G$ is stable, all poles of $G(s)$ are in $\{s \in \mathbb{C} \mid \operatorname{Re} s<0\}$, so that it is holomorphic and bounded at $s=0$. The Laplace transform of the step response is

$$
Y(s)=\frac{G(s)}{s}=\frac{G(s)-G(0)}{s}+\frac{G(0)}{s}=: Y_{\mathrm{tr}}(s)+Y_{\mathrm{ss}}(s) .
$$

Transient response is the signal $y_{\mathrm{tr}}$. Its Laplace transform $Y_{\operatorname{tr}}(s)$ is rational, proper $\left(Y_{\operatorname{tr}}(\infty)=Y(\infty)=0\right)$, and its singularity at $s=0$ is removable, as

$$
\lim _{s \rightarrow 0} Y_{\operatorname{tr}}(s)=\lim _{s \rightarrow 0} \frac{G(s)-G(0)}{s}=G^{\prime}(0)
$$

Hence, $y_{\mathrm{tr}}$ is a superposition of decaying exponents and $\lim _{t \rightarrow \infty} y_{\mathrm{tr}}(t)=0$ meaning that

- the transient response vanishes asymptotically.


## Steady-state and transients of the step responses (contd)

Consider a continuous-time LTI $G$ with a rational transfer function. If $G$ is stable, all poles of $G(s)$ are in $\{s \in \mathbb{C} \mid \operatorname{Re} s<0\}$, so that it is holomorphic and bounded at $s=0$. The Laplace transform of the step response is

$$
Y(s)=\frac{G(s)}{s}=\frac{G(s)-G(0)}{s}+\frac{G(0)}{s}=: Y_{\mathrm{tr}}(s)+Y_{\mathrm{ss}}(s) .
$$

Stead-state response is the scaled step $y_{s s}=G(0) \mathbb{1}$. The constant

- $G(0)$ is called the static gain of $G$.

Thus, the step response of a stable LTI system $G$ converges asymptotically to the step signal scaled by its static gain $G(0)$.

Remark: We refer to $G(0)$ as the static gain of $G$ in the unstable case as well. If $G(0)$ is finite, then we may still think of $y_{s 5}=G(0) \mathbb{1}$ as the steady-state response of an unstable system. However, the transients do not decay then (might even diverge).

## Transients: important characteristics



Smoothness of transients may be measured by the

- overshoot,

$$
\mathrm{OS}:=\frac{y_{\mathrm{peak}}-y_{\mathrm{ss}}}{y_{\mathrm{ss}}}>0 \quad(\text { in } \%)
$$

where $y_{\text {peak }}$ is the highest peak in the direction of $y_{\mathrm{ss}}$,

- undershoot,

$$
\mathrm{US}:=-\frac{y_{\mathrm{us}}}{y_{\mathrm{ss}}}>0 \quad(\text { in } \%)
$$

where $y_{u s}$ is the highest peak against the direction of $y_{\mathrm{ss}}$.

## Steady-state and transients of the step responses (contd)

## Practically,

- steady-state response shows what the response will eventually be
- what is the mercury temperature in a thermometer
- what floor an elevator reaches
- what position the pointer of a spring scale stops
- ...
- transient response shows how the steady state is reached
- how fast a thermometer catch the ambient temperature
- how fast and smooth (comfortable) an elevator moves between floors
- how fast the pointer of a spring scale stops
- how smooth a vehicle responds to road bumps
- how large an electromagnetic pulse is on switching electrical circuits
- ...


## Transients: important characteristics (contd)



Speed of transients may be measured by the

- rise time, $t_{r} \quad$ (time that takes $y$ to rise from $0.1 y_{s s}$ to $0.9 y_{s s}$ )
- peak time, $t_{p}$
(time that takes $y$ to reach its highest peak)
Duration of transients may be measured by the
- settling time, $t_{\mathrm{s}}$ (the smallest $t_{\mathrm{s}}$ such that $\left|\frac{y(t)}{y_{\mathrm{ss}}}-1\right| \leq \frac{\delta}{100}, \forall t \geq t_{\mathrm{s}}$ ) for a given
- settling level $\delta:=\left|\frac{y_{\delta}}{y_{\mathrm{ss}}}\right|$
(in $\%$, most common $\delta=2 \%$ and $\delta=5 \%$ )


## Outline

Step responses of stable 1-order systems

## Step response

If $u=\mathbb{1}$, then

$$
Y(s)=\frac{k_{\mathrm{st}}}{\tau s+1} \frac{1}{s}=k_{\mathrm{st}}\left(\frac{1}{s}-\frac{1}{s+1 / \tau}\right)
$$

and

$$
y(t)=k_{\mathrm{st}}\left(1-\mathrm{e}^{-t / \tau}\right) \mathbb{1}(t),
$$

i.e.

- $y_{\mathrm{ss}}=k_{\mathrm{st}} \mathbb{1}$ is shaped by $k_{\mathrm{st}}$
$-y_{\mathrm{tr}}=-k_{\mathrm{st}}\left(\mathbb{P}_{1 / \tau} \exp _{-1}\right) \mathbb{1}$ is shaped mainly by $\tau \quad k_{\text {st }}$ only scales it

By the initial value theorem,

$$
\dot{y}(0)=\lim _{s \rightarrow \infty} s \cdot s Y(s)=\lim _{s \rightarrow \infty} \frac{k_{\mathrm{st}} s}{\tau s+1}=\frac{k_{\mathrm{st}}}{\tau} \neq 0
$$

## Fundamental 1-order system

An LTI system $G: u \mapsto y$ with the transfer function

$$
G(s)=\frac{k_{\mathrm{st}}}{\tau s+1},
$$

where

- $k_{\mathrm{st}} \neq 0$ is its static gain $\left(k_{\mathrm{st}}=G(0)\right)$
- $\tau>0$ is its time constant


## Examples:

- spring-damper,
- mercury thermometer, , with $\tau \dot{\theta}(t)=\theta_{\mathrm{amb}}(t)-\theta(t)$ has $\tau$ as its time constant and $k_{s t}=1$;
- RL-circuit, ${ }^{2}$, with $\operatorname{Li}(t)+R i(t)=v(t)$ has $\tau=\frac{L}{R} \& k_{\mathrm{st}}=\frac{1}{R}$.


## Step response: transients



This response has

- overshoot OS $=0 \%$ and undershot US $=0 \%$ monotonic
- raise time $t_{r}=(\ln 9) \tau \approx 2.2 \tau$
- settling time $t_{\mathrm{s}}=\left(\ln \frac{100}{\delta}\right) \tau$

$$
y_{\mathrm{tr}}\left(t_{\mathrm{s}}\right)=-\frac{\delta}{100} k_{\mathrm{st}}
$$

Qualitatively, the speed and duration of transients in first-order systems are proportional to the time constant $\tau$, viz.

- the larger $\tau$ is, the slower transients are.


## Outline

Step responses of stable 2-order systems

## Step response

If $u=\mathbb{1}$, then

$$
Y(s)=\frac{k_{s t} \omega_{n}^{2}}{s^{2}+2 \zeta \omega_{n} s+\omega_{n}^{2}} \frac{1}{s}
$$

and the roots of its denominator are

$$
s_{1}=0 \quad \text { and } \quad s_{2,3}=\left(-\zeta \pm \sqrt{\zeta^{2}-1}\right) \omega_{n}
$$

Three cases shall be studied separately

1. if $\zeta>1$, then $s_{2,3}$ are real and simple
2. if $\zeta=1$, then $s_{2,3}$ are real and equal
3. if $\zeta<1$, then $s_{2,3}$ are complex conjugate overdamped critically damped if $\zeta=0$, the

In all cases,

$$
\dot{y}(0)=\lim _{s \rightarrow \infty} s \cdot s Y(s)=\lim _{s \rightarrow \infty} \frac{k_{\mathrm{st}} \omega_{\mathrm{n}}^{2} s}{s^{2}+2 \zeta \omega_{\mathrm{n}} s+\omega_{\mathrm{n}}^{2}}=0
$$

## Fundamental 2-order system

An LTI system $G: u \mapsto y$ with the transfer function

$$
G(s)=\frac{k_{\mathrm{st}} \omega_{\mathrm{n}}^{2}}{s^{2}+2 \zeta \omega_{\mathrm{n}} s+\omega_{\mathrm{n}}^{2}},
$$

where

- $k_{\mathrm{st}} \neq 0$ is its static gain $\left(k_{\mathrm{st}}=G(0)\right)$
- $\omega_{\mathrm{n}}>0$ is its natural frequency
$-\zeta \geq 0$ is its damping factor


## Examples:

- mass-spring-damper, $m$, with $m \ddot{x}(t)+c \dot{x}(t)+k x(t)=f(t)$ has $\omega_{\mathrm{n}}=\sqrt{k / m}, \zeta=0.5 c / \sqrt{k m}$, and $k_{\mathrm{st}}=1 / k ;$
- RLC-circuit, , with $L \ddot{q}(t)+R \dot{q}(t)+C q(t)=v(t)$, where $q$ is the charge, has $\omega_{\mathrm{n}}=\sqrt{C / L}, \zeta=0.5 R / \sqrt{L C}$, and $k_{\mathrm{st}}=1 / C$.


## Step response of overdamped systems

If $\zeta>1$, then

$$
Y(s)=k_{\mathrm{st}}\left(\frac{1}{s}-\frac{\beta}{s-\lambda_{1}}+\frac{\beta-1}{s-\lambda_{2}}\right)
$$

where

$$
\lambda_{1,2}=-\left(\zeta \mp \sqrt{\zeta^{2}-1}\right) \omega_{\mathrm{n}}<0 \quad \text { and } \quad \beta=\frac{1}{2}\left(\frac{\zeta}{\sqrt{\zeta^{2}-1}}+1\right)>1
$$

In the time domain

$$
y(t)=k_{\text {st }}\left(1-\beta \mathrm{e}^{\lambda_{1} t}+(\beta-1) \mathrm{e}^{\lambda_{2} t}\right) \mathbb{1}(t)
$$

with

$$
\dot{y}(t)=k_{\mathrm{st}} \frac{\mathrm{e}^{\lambda_{1} t}-\mathrm{e}^{\lambda_{2} t}}{2 \sqrt{\zeta^{2}-1}} \mathbb{l}(t)>0, \quad \forall t>0
$$

(because $\lambda_{1}>\lambda_{2}$ ), meaning that $y$ is monotonically increasing.
The transient response, $y_{\mathrm{tr}}(t)=-k_{\mathrm{st}} \beta \mathrm{e}^{\lambda_{1} t} \mathbb{\square}(t)+k_{\mathrm{st}}(\beta-1) \mathrm{e}^{\lambda_{2} t} \mathbb{\square}(t)$, is the superposition of transient responses of 1 -order systems.

## Step response of overdamped systems (contd)

Rewrite

$$
y_{\mathrm{tr}}(t)=\overbrace{-k_{\mathrm{st}} \beta \mathrm{e}^{-t / \tau_{1}} \mathbb{1}(t)}^{y_{\mathrm{tr} 1}(t)}+\overbrace{k_{\mathrm{st}}(\beta-1) \mathrm{e}^{-t / \tau_{2}} \mathbb{\mathbb { t }}(t)}^{y_{\mathrm{tr} 2}(t)},
$$

where the time constants and gains are

$$
\tau_{1,2}=\frac{\zeta \pm \sqrt{\zeta^{2}-1}}{\omega_{\mathrm{n}}}={\underset{1}{2}}_{0_{1}}^{\substack{\tau_{1,2} \omega_{n} \\ 2}} \underbrace{}_{\zeta} \quad \text { and } \quad \beta=\frac{\zeta+\sqrt{\zeta^{2}-1}}{2 \sqrt{\zeta^{2}-1}}=\underbrace{\beta}_{1-1.5} \underbrace{}_{1.5}
$$

## As $\zeta$ grows,

$-\tau_{1}$ grows with respect to $\tau_{2} \Longrightarrow y_{\mathrm{tr} 2}$ decays faster than $y_{\mathrm{tr} 1}$
$-\beta$ grows with respect to $\beta-1 \Longrightarrow y_{\mathrm{tr} 2}$ becomes smaller than $y_{\mathrm{tr} 1}$
like in

meaning that $y_{\mathrm{tr} 2}$ may be neglected for large $\zeta$ 's, say for $\zeta \geq 1.25$.

## Step response of over- and critically damped systems



$$
\begin{aligned}
& t_{\mathrm{r}} \approx \frac{1}{\omega_{\mathrm{n}}} \begin{cases}4.9 \zeta-1.5 & \text { if } 1 \leq \zeta<2 \\
4.4 \zeta-0.5 & \text { if } \zeta \geq 2\end{cases} \\
& t_{\mathrm{s}, 5} \approx \frac{1}{\omega_{\mathrm{n}}} \begin{cases}-0.6 \zeta^{2}+8.5 \zeta-3.1 & \text { if } 1 \leq \zeta<2 \\
6 \zeta-0.5 & \text { if } \zeta \geq 2\end{cases} \\
& t_{\mathrm{s}, 2} \approx \frac{1}{\omega_{\mathrm{n}}} \begin{cases}-1.1 \zeta^{2}+12.3 \zeta-5.3 & \text { if } 1 \leq \zeta<2 \\
7.9 \zeta-0.9 & \text { if } \zeta \geq 2\end{cases}
\end{aligned}
$$

This response has

- overshoot $\mathrm{OS}=0 \%$ and undershot US $=0 \%$
- raise time $t_{r}$
- monotonically increases with $\zeta$
- is inversely proportional to $\omega_{\mathrm{n}}$
- settling time $t_{\mathrm{s}}$
- monotonically increases with $\zeta$
- is inversely proportional to $\omega_{\mathrm{n}}$


## Step response of critically damped systems

If $\zeta=1$, then

$$
Y(s)=k_{\mathrm{st}}\left(\frac{1}{s}-\frac{1}{s+\omega_{\mathrm{n}}}-\frac{\omega_{\mathrm{n}}}{\left(s+\omega_{\mathrm{n}}\right)^{2}}\right)
$$

In the time domain

$$
y(t)=k_{\mathrm{st}}\left(1-\left(1+\omega_{\mathrm{n}} t\right) \mathrm{e}^{-\omega_{\mathrm{n}} t}\right) \mathbb{1}(t)
$$

with

$$
\dot{y}(t)=k_{\mathrm{st}} \omega_{\mathrm{n}}^{2} t \mathrm{e}^{-\omega_{\mathrm{n}} t} \mathbb{1}(t)>0, \quad \forall t>0
$$

meaning that $y$ is monotonically increasing in this case as well.

## Step response of underdamped systems

If $0 \leq \zeta<1$, then

$$
\begin{aligned}
Y(s) & =k_{\mathrm{st}}\left(\frac{1}{s}-\frac{s+2 \zeta \omega_{\mathrm{n}}}{s^{2}+2 \zeta \omega_{\mathrm{n}} s+\omega_{\mathrm{n}}^{2}}\right)=k_{\mathrm{st}}\left(\frac{1}{s}-\frac{\left(s+\zeta \omega_{\mathrm{n}}\right)+\zeta \omega_{\mathrm{n}}}{\left(s+\zeta \omega_{\mathrm{n}}\right)^{2}+\left(1-\zeta^{2}\right) \omega_{\mathrm{n}}^{2}}\right) \\
& =k_{\mathrm{st}}\left(\frac{1}{s}-\frac{1}{\sqrt{1-\zeta^{2}}} \frac{\sqrt{1-\zeta^{2}}\left(s+\zeta \omega_{\mathrm{n}}\right)+\zeta \omega_{\mathrm{d}}}{\left(s+\zeta \omega_{\mathrm{n}}\right)^{2}+\omega_{\mathrm{d}}^{2}}\right)
\end{aligned}
$$

where $\omega_{\mathrm{d}}:=\sqrt{1-\zeta^{2}} \omega_{\mathrm{n}}$ is the damped natural frequency. Hence,

$$
y(t)=k_{\mathrm{st}}\left(1-\frac{1}{\sqrt{1-\zeta^{2}}} \mathrm{e}^{-\zeta \omega_{\mathrm{n}} t} \sin \left(\omega_{\mathrm{d}} t+\arccos \zeta\right)\right) \mathbb{1}(t)
$$

and it is always within the envelope

$$
k_{\mathrm{st}}\left(1-\frac{1}{\sqrt{1-\zeta^{2}}} \mathrm{e}^{-\zeta \omega_{\mathrm{n}} t}\right) \mathbb{1}(t) \leq y(t) \leq k_{\mathrm{st}}\left(1+\frac{1}{\sqrt{1-\zeta^{2}}} \mathrm{e}^{-\zeta \omega_{\mathrm{n}} t}\right) \mathbb{1}(t)
$$

Step response of underdamped systems: transients
$k_{\text {st }}\left(1+\mathrm{e}^{-\pi \xi / \sqrt{1-\xi^{2}}}\right)$.


$$
\begin{aligned}
& t_{\mathrm{r}} \approx \frac{1.6 \zeta^{3}-0.17 \zeta^{2}+0.92 \zeta+1.02}{\omega_{\mathrm{n}}} \\
& t_{5,5} \approx \frac{2.88}{\left(-\zeta^{2}+0.64 \zeta+0.96\right) \zeta \omega_{\mathrm{n}}} \\
& t_{\mathrm{s}, 2} \approx \frac{6.49}{\left(-\zeta^{2}+0.45 \zeta+1.65\right) \zeta \omega_{\mathrm{n}}}
\end{aligned}
$$

This response has

- overshoot $\mathrm{OS}=\mathrm{e}^{-\pi \zeta / \sqrt{1-\zeta^{2}}} \cdot 100 \%$ and undershot US $=0 \%$
- raise time $t_{r}$ is inversely proportional to $\omega_{\mathrm{n}}$
- settling time $t_{s}$ is inversely proportional to $\omega_{\mathrm{n}}$

Qualitatively,

- the smaller $\zeta$ is, the more oscillatory the response is
(larger overshoot, up to $\mathrm{OS}=100 \%$ for $\zeta=0$, and longer $t_{\mathrm{s}}$ ).

Step response of underdamped systems: $\alpha>0$


## As $\alpha$ grows,

- the overshoot OS increases
- the raise time $t_{r}$ decreases
- the settling time $t_{\mathrm{s}}$ increases


## Step response of underdamped systems: effect of zeros

Let

$$
G_{\alpha}(s)=\frac{k_{\mathrm{st}}\left(\alpha \omega_{\mathrm{n}} s+\omega_{\mathrm{n}}^{2}\right)}{s^{2}+2 \zeta \omega_{\mathrm{n}} s+\omega_{\mathrm{n}}^{2}},
$$

for $\alpha \in \mathbb{R}$. This transfer function is said to have a zero at $s=-\omega_{\mathrm{n}} / \alpha$ since $G_{\alpha}\left(-\omega_{\mathrm{n}} / \alpha\right)=0$. In this case

$$
Y_{\alpha}(s)=\frac{k_{\mathrm{st}}\left(\alpha \omega_{\mathrm{n}} s+\omega_{\mathrm{n}}^{2}\right)}{s^{2}+2 \zeta \omega_{\mathrm{n}} s+\omega_{\mathrm{n}}^{2}} \frac{1}{s}=Y_{0}(s)+\frac{\alpha}{\omega_{\mathrm{n}}} s Y_{0}(s)
$$

and

$$
y_{\alpha}(t)=y_{0}(t)+\frac{\alpha}{\omega_{\mathrm{n}}} \dot{y}_{0}(t)
$$

where $y_{0}$ is the response with $\alpha=0$ (no zeros) and

$$
\frac{\alpha}{\omega_{\mathrm{n}}} \dot{y}_{0}(t)=\frac{\alpha}{\sqrt{1-\zeta^{2}}} \sin \left(\omega_{\mathrm{d}} t\right)
$$

Step response of underdamped systems: $\alpha<0$


As $\alpha$ decreases,

- the overshoot OS increases
- the undershoot US increases
- the raise time $t_{r}$ decreases
- the settling time $t_{\mathrm{s}}$ increases

