

# Linear Systems (034032)

## lecture no. 7

Leonid Mirkin

Faculty of Mechanical Engineering  
Technion—IIT



# Outline

Transfer functions of LTI systems

LTI systems with rational transfer functions

Steady-state and transient responses

Step responses of stable 1-order systems

Step responses of stable 2-order systems

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## Transfer functions of LTI systems

LTI systems with rational transfer functions

Steady-state and transient responses

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## LTI systems response: from time to Laplace / $z$ domain

We know that if  $G : u \mapsto y$  is LTI (linear time invariant), then

$$y = g * u$$

where  $g = G\delta$  is the impulse response of  $G$ , i.e. its response to  $\delta$  applied at  $t = 0$ . By the convolution property of the Laplace /  $z$  transform,

$$Y(s) = G(s)U(s) \quad \text{or} \quad Y(z) = G(z)U(z),$$

In other words, (dynamic) LTI systems in the Laplace /  $z$  domain act as the (static) multiplication of the transformed impulse response and input.

$$G(s) = (\mathcal{L}\{g\})(s) \quad \text{or} \quad G(z) = (\mathcal{Z}\{g\})(z)$$

is called the transfer function of  $G$ . Transfer function may also be viewed as the ratio of the Laplace /  $z$  transforms of the output and input signals,

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## Transfer functions: examples

**Gain:** its impulse response  $g = k\delta$ , so  $G(s) = k$  and  $G(z) = k$ .

**Delay:** its impulse response  $g = \delta_{-r}$ , so  $G(s) = e^{-rs}$  and  $G(z) = z^{-r}$ .

**Integrator:** its impulse response  $g_{\text{int}} = 1$ , so

$$G_{\text{int}}(s) = \frac{1}{s} \quad \text{and} \quad G_{\text{int}}(z) = \frac{z}{z-1}.$$

has a single pole at  $s = 0$  and  $z = 1$ , respectively.

**Finite-memory integrator:** its impulse response  $g_{\text{fmi},\mu} = \delta_{-\mu/2} \text{rect}_{\mu}$ , so

$$G_{\text{fmi},\mu}(s) = \frac{1 - e^{-\mu s}}{s} = \int_0^{\mu} e^{-st} dt$$

is an entire function.

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## Transfer function description

The fact that *dynamic* LTI systems can be described by *algebraic* relations,

$$Y(s) = G(s)U(s) \quad \text{or} \quad Y(z) = G(z)U(z),$$

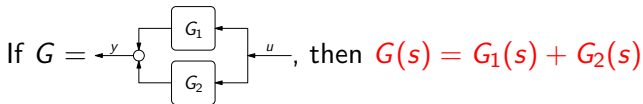
is a handy property, having a potential of greatly simplifying the analysis. It has no counterparts in the realm of general time-varying/nonlinear systems, making the class of LTI systems so special and sought after.

Still, we yet to know how to

- express properties of LTI systems in terms of their transfer functions, without which algebraic relations are not more than a curiosity.

# System interconnections via transfer functions

Parallel:



sum

Cascade: If  $G =$

, then  $G(s) = G_2(s)G_1(s)$

product

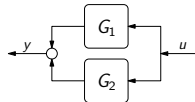
Feedback: If  $G =$

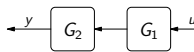
, then  $G(s) = \frac{G_1(s)}{1 + G_1(s)G_2(s)}$

or

$$Y = G_1(s)U + G_2(s)Y \Leftrightarrow (1 - G_1(s)G_2(s))Y = G_1(s)U$$

## System interconnections via transfer functions

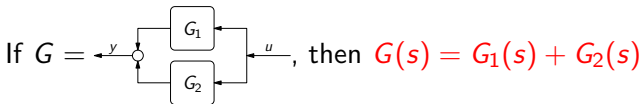
Parallel: If  $G =$  , then  $G(s) = G_1(s) + G_2(s)$  sum

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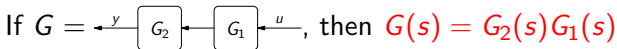
# System interconnections via transfer functions

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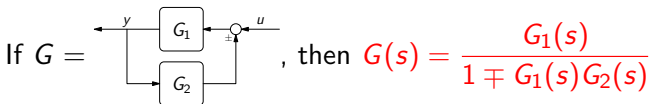
sum

Cascade:



product

Feedback:



by

$$Y = G_1(U \pm G_2 Y) \iff (1 \mp G_1 G_2) Y = G_1 U$$

## Causality via transfer functions: discrete systems

### Theorem

Let  $G$  be an LTI discrete-time system, whose transfer function  $G(z)$  has its RoC  $\subset \{z \in \mathbb{C} \mid |z| > \alpha_g\}$  for some  $\alpha_g \in \mathbb{R}_+$ . Such  $G$  is causal iff

$$\limsup_{|z| \rightarrow \infty} |G(z)| < \infty.$$

*Proof (outline):*  $G$  is causal iff  $\text{supp}(g) \subset \mathbb{Z}_+$ . The transfer function

$$G(z) = (\mathcal{Z}\{g\})(z) = \sum_{t \in \mathbb{Z}} g[t]z^{-t} = \sum_{t=-\infty}^{-1} g[t]z^{-t} + \sum_{t=0}^{\infty} g[t]z^{-t}$$

If  $|z| \rightarrow \infty$ , then  $|z^{-t}| \rightarrow \infty$  for every  $t \in \mathbb{Z} \setminus \mathbb{Z}_+$  as well and the first term above remains bounded only if  $\text{supp}(g) \subset \mathbb{Z}_+$ . If  $\text{supp}(g) \subset \mathbb{Z}_+$ , then the second term is in the RoC, so  $|G(z)|$  is bounded, if  $|z|$  is large enough.  $\square$

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**Remark:** There is no similar result for continuous-time systems. For example,  $se^{-s}$  is not bounded if  $\text{Re } s \rightarrow +\infty$ , but it corresponds to the causal system acting as  $y(t) = \dot{u}(t-1)$ .



## $L_2 / \ell_2$ stability via transfer functions

### Theorem

*A continuous-time LTI system is causal and  $L_2$ -stable iff its transfer function is holomorphic and bounded in  $\mathbb{C}_0 := \{s \in \mathbb{C} \mid \operatorname{Re} s > 0\}$ .*

### Theorem

*A discrete-time LTI system is causal and  $\ell_2$ -stable iff its transfer function is holomorphic and bounded in  $\{z \in \mathbb{C} \mid |z| > 1\}$ .*

Example 1: we know that  $G_{\text{rml},\mu}(s) = \int_0^\mu e^{-st} dt$ , which is entire. Now,

$$|G_{\text{rml},\mu}(s)| = \left| \int_0^\mu e^{-st} dt \right| \leq \int_0^\mu |e^{-st}| dt = \int_0^\mu e^{-\operatorname{Re} s t} dt < \int_0^\mu dt = \mu$$

for all  $\operatorname{Re} s > 0$ . Thus,  $G_{\text{rml},\mu}(s)$  is not only holomorphic, but also bounded in  $\mathbb{C}_0$ , meaning that  $G_{\text{rml},\mu}$  is  $L_2$ -stable.

Example 2: Consider  $G : u \mapsto y$  acting as  $y(t) = \dot{u}(t)$ . Its transfer function  $G(s) = s$  is entire, but not bounded in  $\mathbb{C}_0$ , meaning that  $G$  is not  $L_2$ -stable.

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## LTI systems described by ODE

Let  $G : u \mapsto y$  be an analog LTI system described as

$$\begin{aligned}y^{(n)}(t) + a_{n-1}y^{(n-1)}(t) + \cdots + a_1\dot{y}(t) + a_0y(t) \\ = b_m u^{(m)}(t) + b_{m-1}u^{(m-1)}(t) + \cdots + b_1\dot{u}(t) + b_0u(t)\end{aligned}$$

for some  $a_i, b_i \in \mathbb{R}$ . In the Laplace domain this relation reads

$$Y(s) = \frac{b_m s^m + b_{m-1} s^{m-1} + \cdots + b_1 s + b_0}{s^n + a_{n-1} s^{n-1} + \cdots + a_1 s + a_0} U(s).$$

Hence, the transfer function of this system is

$$G(s) = \frac{b_m s^m + b_{m-1} s^{m-1} + \cdots + b_1 s + b_0}{s^n + a_{n-1} s^{n-1} + \cdots + a_1 s + a_0}.$$

In other words,

- transfer functions of LTI systems described by ODE are always rational (actually, real-rational if the coefficients are real).

## LTI systems described by difference equations

Let  $G : u \mapsto y$  be a discrete LTI system described as

$$\begin{aligned}y[t + n] + a_{n-1}y[t + n - 1] + \cdots + a_1y[t + 1] + a_0y[t] \\ = b_m[t + m] + b_{m-1}u[t + m - 1] + \cdots + b_1u[t + 1] + b_0u[t]\end{aligned}$$

for some  $a_i, b_i \in \mathbb{R}$ . In the Laplace domain this relation reads

$$Y(z) = \frac{b_m z^m + b_{m-1} z^{m-1} + \cdots + b_1 z + b_0}{z^n + a_{n-1} z^{n-1} + \cdots + a_1 z + a_0} U(z).$$

Hence, the transfer function of this system is

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again real-rational.

## Interconnections

If

$$G_1(s) = \frac{N_1(s)}{D_1(s)} \quad \text{and} \quad G_2(s) = \frac{N_2(s)}{D_2(s)},$$

then

$$G_1(s) + G_2(s) = \frac{N_1(s)}{D_1(s)} + \frac{N_2(s)}{D_2(s)} = \frac{N_1(s)D_2(s) + N_2(s)D_1(s)}{D_1(s)D_2(s)}$$

$$G_2(s)G_1(s) = \frac{N_2(s)}{D_2(s)} \frac{N_1(s)}{D_1(s)} = \frac{N_2(s)N_1(s)}{D_2(s)D_1(s)}$$

$$\begin{aligned} \frac{G_1(s)}{1 \mp G_1(s)G_2(s)} &= \frac{\frac{N_1(s)}{D_1(s)}}{1 \mp \frac{N_1(s)}{D_1(s)} \frac{N_2(s)}{D_2(s)}} = \frac{\frac{N_1(s)}{D_1(s)}}{\frac{D_1(s)D_2(s) \mp N_1(s)N_2(s)}{D_1(s)D_2(s)}} \\ &= \frac{N_1(s)D_2(s)}{D_1(s)D_2(s) \mp N_1(s)N_2(s)} \end{aligned}$$

are rational as well. In other words,

parallel, cascade, & feedback interconnections of systems with rational transfer functions result in systems with rational transfer functions.

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are rational as well. In other words,

- parallel, cascade, & feedback interconnections of systems with rational transfer functions result in systems with rational transfer functions.



## Impulse response of continuous-time systems

Let  $G$  be an LTI system with a rational proper transfer function of the form

$$G(s) = \frac{b_m s^m + b_{m-1} s^{m-1} + \dots + b_1 s + b_0}{s^n + a_{n-1} s^{n-1} + \dots + a_1 s + a_0}, \quad n \geq m$$

Let it have  $k$  distinct poles  $p_i \in \mathbb{C}$  of order  $n_i \in \mathbb{N}$ . In this case

$$G(s) = G(\infty) + \sum_{i=1}^k \sum_{j=1}^{n_i} \frac{c_{ij}}{(s - p_i)^j}$$

The inverse Laplace transforms  $g_{ij}$  of  $G_{ij}(s) := c_{ij}/(s - p_i)^j$  are

$$g_{i1}(t) = c_{i1} e^{p_i t} \mathbb{1}(t) \quad \text{and} \quad g_{ij}(t) = \frac{c_{ij}}{(j-1)!} t^{j-1} e^{p_i t} \mathbb{1}(t), \quad j > 1$$

(by the  $t$ -modulation property). Therefore, by linearity

$$g(t) = G(\infty) \delta(t) + \sum_{i=1}^k \left( c_{i1} + \sum_{j=2}^{n_i} \frac{c_{ij}}{(j-1)!} t^{j-1} \right) e^{p_i t} \mathbb{1}(t),$$

i.e. it is a superposition of ( $t$ -modulated) **exponential** functions.

## Impulse response of continuous-time systems (contd)

If

- $\operatorname{Re} p_i \geq 0$ , then  $\sum_{j=1}^{n_i} g_{ij} \notin L_1$  (terms do not decay)
- $\operatorname{Re} p_i < 0$ , then

$$\int_0^{\infty} \left| \frac{c_{ij}}{(j-1)!} t^{j-1} e^{p_i t} \right| dt = \frac{|c_{ij}|}{(-\operatorname{Re} p_i)^j} \implies \sum_{j=1}^{n_i} g_{ij} \in L_1$$

If  $\operatorname{Im} p_h \neq 0$ , then  $\exists h_2 \neq h_1$  such that  $p_{h_2} = \bar{p}_{h_1}$  and also  $c_{h_2 1} = \bar{c}_{h_1 1}$ . Then

$$\begin{aligned} \frac{c_{h_1 1}}{s - p_{h_1}} + \frac{c_{h_2 1}}{s - p_{h_2}} &= \frac{c_{h_1 1}}{s - p_{h_1}} + \frac{\bar{c}_{h_1 1}}{s - \bar{p}_{h_1}} = \frac{c_{h_1 1}(s - \bar{p}_{h_1}) + \bar{c}_{h_1 1}(s - p_{h_1})}{(s - p_{h_1})(s - \bar{p}_{h_1})} \\ &= \frac{2 \operatorname{Re} c_{h_1 1}(s - \operatorname{Re} p_{h_1}) - 2 \operatorname{Im} c_{h_1 1} \operatorname{Im} p_{h_1}}{(s - \operatorname{Re} p_{h_1})^2 + (\operatorname{Im} p_{h_1})^2} \end{aligned}$$

Thus, if all poles are simple, then we can always end up with an alternative partial fraction expansion with 1- and 2-order real-rational terms only. And we know that they correspond to exponentials and (modulated) sine waves.

## Impulse response of continuous-time systems (contd)

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$$\int_0^{\infty} \left| \frac{c_{ij}}{(j-1)!} t^{j-1} e^{p_i t} \right| dt = \frac{|c_{ij}|}{(-\operatorname{Re} p_i)^j} \implies \sum_{j=1}^{n_i} g_{ij} \in L_1$$

If  $\operatorname{Im} p_{i_1} \neq 0$ , then  $\exists i_2 \neq i_1$  such that  $p_{i_2} = \overline{p_{i_1}}$  and also  $c_{i_2 1} = \overline{c_{i_1 1}}$ . Then

$$\begin{aligned} \frac{c_{i_1 1}}{s - p_{i_1}} + \frac{c_{i_2 1}}{s - p_{i_2}} &= \frac{c_{i_1 1}}{s - p_{i_1}} + \frac{\overline{c_{i_1 1}}}{s - \overline{p_{i_1}}} = \frac{c_{i_1 1}(s - \overline{p_{i_1}}) + \overline{c_{i_1 1}}(s - p_{i_1})}{(s - p_{i_1})(s - \overline{p_{i_1}})} \\ &= \frac{2 \operatorname{Re} c_{i_1 1}(s - \operatorname{Re} p_{i_1}) - 2 \operatorname{Im} c_{i_1 1} \operatorname{Im} p_{i_1}}{(s - \operatorname{Re} p_{i_1})^2 + (\operatorname{Im} p_{i_1})^2} \end{aligned}$$

Thus, if all poles are simple, then we can always end up with an alternative partial fraction expansion with 1- and 2-order real-rational terms only. And we know that they correspond to exponentials and (modulated) sine waves.

## Impulse response of continuous-time systems: example

Consider

$$\begin{aligned}
 G(s) &= \frac{s^2 - 2s + 6}{s^4 + 5s^3 + 8s^2 + 6s} = \frac{s^2 - 2s + 6}{s(s+3)(s^2 + 2s + 2)} \\
 &= \frac{\text{Res}(G, 0)}{s} + \frac{\text{Res}(G, -3)}{s+3} + \frac{\text{Res}(G, -1+j)}{s+1-j} + \frac{\text{Res}(G, -1-j)}{s+1+j} \\
 &= \frac{1}{s} - \frac{1.4}{s+3} + \frac{0.2+1.4j}{s+1-j} + \frac{0.2-1.4j}{s+1+j} \\
 &= \frac{1}{s} - \frac{1.4}{s+3} + \frac{0.4s-2.8}{s^2+2s+2} = \frac{1}{s} - \frac{1.4}{s+3} + 2\sqrt{2} \frac{\frac{\sqrt{2}}{10}(s+1) - \frac{7\sqrt{2}}{10}}{(s+1)^2+1}
 \end{aligned}$$

Hence, taking the inverse Laplace transform, we obtain  $g(t) = 2.00 \cos(2t) + 3.11 \sin(2t) + 1.4e^{-3t} - 1.4e^{-t} \cos(t+3) + 2.82e^{-t} \sin(t+3)$

$$g(t) = 1(t) - 1.4e^{-3t}1(t) + 2\sqrt{2}e^{-t} \sin(t+3)1(t)$$

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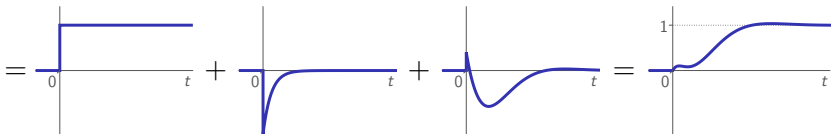
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 \end{aligned}$$

Hence, taking into account that  $\arccos(-0.7\sqrt{2}) = 2.999695599 \approx 3$ ,

$$g(t) = \mathbb{1}(t) - 1.4e^{-3t}\mathbb{1}(t) + 2\sqrt{2}e^{-t}\sin(t+3)\mathbb{1}(t)$$



## Impulse response of discrete-time systems

Rational transfer functions in the  $z$ -domain are also decomposed to partial fractions. Time responses are then also superposition of exponential terms. However, discrete-time responses of corresponding terms are

- way messier than their continuous-time counterparts and less intuitive, with some additional properties, like possibly oscillatory response of 1-order terms. For those reasons we won't discuss details.

## I/O stability via transfer functions

### Theorem

*If the transfer function  $G(s)$  of a continuous-time LTI system  $G$  is rational, then  $G$  is causal and I/O stable iff*

- $G(s)$  is proper and has no poles in  $\bar{\mathbb{C}}_0 := \{s \in \mathbb{C} \mid \operatorname{Re} s \geq 0\}$

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*Example 2 (contd):* The differentiator  $y(t) = \dot{u}(t)$  has the rational transfer function  $G(s) = s$ . But it is not proper, attesting to what we already saw.

*Example 3:* The discrete system  $G: u \mapsto y$  acting as  $y[t] = u[t+4] - u[t]$  has the (rational) transfer function  $G(z) = z^4 - 1$ . This  $G(z)$  is non-proper and has no poles. Hence,  $G$  is not causal, but is stable.



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Transfer functions of LTI systems

LTI systems with rational transfer functions

**Steady-state and transient responses**

Step responses of stable 1-order systems

Step responses of stable 2-order systems

## Persistent test signals

Responses of systems obviously vary depending on inputs. It is not realistic to have a universal understanding of responses of dynamic systems to every possible input. Instead, we shall try to understand responses to (relatively) simple signals, which may nevertheless be *representative* in many situations (unlike the impulse). Such signals, termed **test signals**, normally

- have support in  $\mathbb{R}_+$  (so the action starts at  $t = 0$ )
- are persistent (i.e. do not decay)
- have some regularity (like convergent or periodic)

Some commonly used test signals:

- step  $u(t) = 1(t)$   
represents a quick change of an otherwise constant or slowly varying environment
- sine wave  $u(t) = \sin(\omega_u t + \phi_u) 1(t)$   
 $T$ -periodic

with only pure imaginary poles as singularities of their Laplace transforms.

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with only **pure imaginary poles** as singularities of their Laplace transforms.

## Steady-state and transients of the step responses

Consider a continuous-time LTI  $G$  with a rational transfer function. If  $G$  is stable, all poles of  $G(s)$  are in  $\{s \in \mathbb{C} \mid \operatorname{Re} s < 0\}$ , so that it is holomorphic and bounded at  $s = 0$ . The Laplace transform of the step response is

$$Y(s) = \frac{G(s)}{s} = \frac{G(s) - G(0)}{s} + \frac{G(0)}{s} =: Y_{\text{tr}}(s) + Y_{\text{ss}}(s).$$

Transient response is the signal  $y_{\text{tr}}$ . Its Laplace transform  $Y_{\text{tr}}(s)$  is rational, proper ( $Y_{\text{tr}}(\infty) = Y(\infty) = 0$ ), and its singularity at  $s = 0$  is removable, as

$$\lim_{s \rightarrow 0} Y_{\text{tr}}(s) = \lim_{s \rightarrow 0} \frac{G(s) - G(0)}{s} = G'(0).$$

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Thus, the step response of a stable LTI system  $G$  converges asymptotically to the step signal scaled by its static gain  $G(0)$ .



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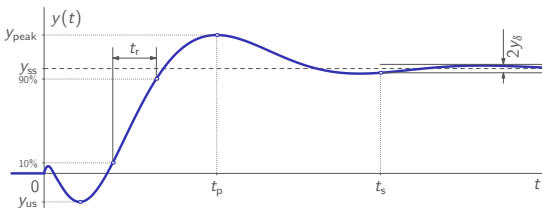
**Remark:** We refer to  $G(0)$  as the static gain of  $G$  in the unstable case as well. If  $G(0)$  is finite, then we may still think of  $y_{\text{ss}} = G(0)\mathbb{1}$  as the steady-state response of an unstable system. However, the transients do not decay then (might even diverge).

## Steady-state and transients of the step responses (contd)

Practically,

- steady-state response shows **what** the response will eventually be
  - what is the mercury temperature in a thermometer
  - what floor an elevator reaches
  - what position the pointer of a spring scale stops
  - ...
- transient response shows **how** the steady state is reached
  - how fast a thermometer catch the ambient temperature
  - how fast and smooth (comfortable) an elevator moves between floors
  - how fast the pointer of a spring scale stops
  - how smooth a vehicle responds to road bumps
  - how large an electromagnetic pulse is on switching electrical circuits
  - ...

## Transients: important characteristics



Smoothness of transients may be measured by the

- overshoot,

$$OS := \frac{y_{peak} - y_{ss}}{y_{ss}} > 0 \quad (\text{in } \%),$$

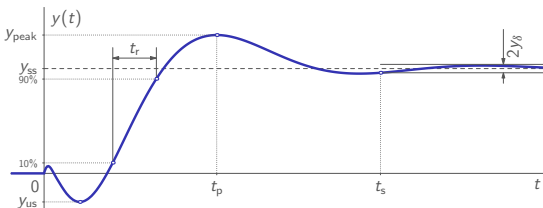
where  $y_{peak}$  is the highest peak in the direction of  $y_{ss}$ ,

- undershoot,

$$US := -\frac{y_{us}}{y_{ss}} > 0 \quad (\text{in } \%),$$

where  $y_{us}$  is the highest peak against the direction of  $y_{ss}$ .

## Transients: important characteristics (contd)



Speed of transients may be measured by the

- rise time,  $t_r$  (time that takes  $y$  to rise from  $0.1y_{ss}$  to  $0.9y_{ss}$ )
- peak time,  $t_p$  (time that takes  $y$  to reach its highest peak)

Duration of transients may be measured by the

- settling time,  $t_s$  (the smallest  $t_s$  such that  $\left| \frac{y(t)}{y_{ss}} - 1 \right| \leq \frac{\delta}{100}, \forall t \geq t_s$ )

for a given

- settling level  $\delta := \left| \frac{y\delta}{y_{ss}} \right|$  (in %, most common  $\delta = 2\%$  and  $\delta = 5\%$ )

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## Fundamental 1-order system

An LTI system  $G : u \mapsto y$  with the transfer function

$$G(s) = \frac{k_{\text{st}}}{\tau s + 1},$$

where

- $k_{\text{st}} \neq 0$  is its static gain ( $k_{\text{st}} = G(0)$ )
- $\tau > 0$  is its **time constant**

Examples:

- spring-damper, , with  $c\dot{x}(t) + kx(t) = f(t)$  has  $\tau = \frac{c}{k}$  &  $k_{\text{st}} = \frac{1}{k}$ ;
- mercury thermometer, , with  $\tau\dot{\theta}(t) = \theta_{\text{amb}}(t) - \theta(t)$  has  $\tau$  as its time constant and  $k_{\text{st}} = 1$ ;
- RL-circuit, , with  $L\dot{i}(t) + Ri(t) = v(t)$  has  $\tau = \frac{L}{R}$  &  $k_{\text{st}} = \frac{1}{R}$ .

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


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## Step response

If  $u = \mathbb{1}$ , then

$$Y(s) = \frac{k_{st}}{\tau s + 1} \frac{1}{s} = k_{st} \left( \frac{1}{s} - \frac{1}{s + 1/\tau} \right)$$

and

$$y(t) = k_{st}(1 - e^{-t/\tau})\mathbb{1}(t),$$

i.e.

- $y_{ss} = k_{st}\mathbb{1}$  is shaped by  $k_{st}$
- $y_{tr} = -k_{st}(\mathbb{P}_{1/\tau} \exp_{-1})\mathbb{1}$  is shaped mainly by  $\tau$   $k_{st}$  only scales it

By the initial value theorem,

$$\dot{y}(0) = \lim_{s \rightarrow \infty} s \cdot sY(s) = \lim_{s \rightarrow \infty} \frac{k_{st}s}{\tau s + 1} = \frac{k_{st}}{\tau} \neq 0$$



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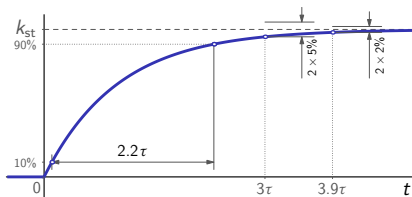
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## Step response: transients



This response has

- overshoot  $OS = 0\%$  and undershoot  $US = 0\%$
- raise time  $t_r = (\ln 9)\tau \approx 2.2\tau$
- settling time  $t_s = (\ln \frac{100}{\delta})\tau$

monotonic

$$y_{tr}(t_s) = -\frac{\delta}{100} k_{st}$$

Qualitatively, the speed and duration of transients in first-order systems are proportional to the time constant  $\tau$ , viz.

- the larger  $\tau$  is, the slower transients are.

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An LTI system  $G : u \mapsto y$  with the transfer function

$$G(s) = \frac{k_{\text{st}}\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2},$$

where

- $k_{\text{st}} \neq 0$  is its static gain ( $k_{\text{st}} = G(0)$ )
- $\omega_n > 0$  is its **natural frequency**
- $\zeta \geq 0$  is its **damping factor**

Examples:

- mass-spring-damper,  $\ddot{x}(t) + c\dot{x}(t) + kx(t) = f(t)$  has  $\omega_n = \sqrt{k/m}$ ,  $\zeta = 0.5c/\sqrt{km}$ , and  $k_{\text{st}} = 1/k$ ;
- RLC-circuit,  $L\ddot{q}(t) + R\dot{q}(t) + Cq(t) = v(t)$ , where  $q$  is the charge, has  $\omega_n = \sqrt{C/L}$ ,  $\zeta = 0.5R/\sqrt{LC}$ , and  $k_{\text{st}} = 1/C$ .

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

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## Step response

If  $u = \mathbb{1}$ , then

$$Y(s) = \frac{k_{st}\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \frac{1}{s}$$

and the roots of its denominator are

$$s_1 = 0 \quad \text{and} \quad s_{2,3} = (-\zeta \pm \sqrt{\zeta^2 - 1})\omega_n$$

Three cases shall be studied separately

1. if  $\zeta > 1$ , then  $s_{2,3}$  are real and simple overdamped
  2. if  $\zeta = 1$ , then  $s_{2,3}$  are real and equal critically damped
  3. if  $\zeta < 1$ , then  $s_{2,3}$  are complex conjugate underdamped
- if  $\zeta = 0$ , then we say that the system is *undamped*; no need in a separate analysis

In all cases,

$$y(0) = \lim_{s \rightarrow \infty} s \cdot sY(s) = \lim_{s \rightarrow \infty} \frac{k_{st}\omega_n^2 s}{s^2 + 2\zeta\omega_n s + \omega_n^2} = 0$$

## Step response

If  $u = \mathbb{1}$ , then

$$Y(s) = \frac{k_{st}\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \frac{1}{s}$$

and the roots of its denominator are

$$s_1 = 0 \quad \text{and} \quad s_{2,3} = (-\zeta \pm \sqrt{\zeta^2 - 1})\omega_n$$

Three cases shall be studied separately

1. if  $\zeta > 1$ , then  $s_{2,3}$  are real and simple overdamped
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## Step response of overdamped systems

If  $\zeta > 1$ , then

$$Y(s) = k_{st} \left( \frac{1}{s} - \frac{\beta}{s - \lambda_1} + \frac{\beta - 1}{s - \lambda_2} \right)$$

where

$$\lambda_{1,2} = -(\zeta \mp \sqrt{\zeta^2 - 1})\omega_n < 0 \quad \text{and} \quad \beta = \frac{1}{2} \left( \frac{\zeta}{\sqrt{\zeta^2 - 1}} + 1 \right) > 1$$

In the time domain

$$y(t) = k_{st} (1 - \beta e^{\lambda_1 t} + (\beta - 1) e^{\lambda_2 t}) \mathbb{1}(t)$$

with

$$\dot{y}(t) = k_{st} \frac{e^{\lambda_1 t} - e^{\lambda_2 t}}{2\sqrt{\zeta^2 - 1}} \mathbb{1}(t) > 0, \quad \forall t > 0$$

(because  $\lambda_1 > \lambda_2$ ), meaning that  $y$  is monotonically increasing.

The transient response,  $y_t(t) = -k_{st}\beta e^{\lambda_1 t} \mathbb{1}(t) + k_{st}(\beta - 1)e^{\lambda_2 t} \mathbb{1}(t)$ , is the superposition of transient responses of 1-order systems.



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## Step response of overdamped systems (contd)

Rewrite

$$y_{\text{tr}}(t) = \overbrace{-k_{\text{st}}\beta e^{-t/\tau_1} \mathbb{1}(t)}^{y_{\text{tr1}}(t)} + \overbrace{k_{\text{st}}(\beta - 1)e^{-t/\tau_2} \mathbb{1}(t)}^{y_{\text{tr2}}(t)},$$

where the time constants and gains are

$$\tau_{1,2} = \frac{\zeta \pm \sqrt{\zeta^2 - 1}}{\omega_n} = \begin{array}{c} \tau_{1,2}\omega_n \\ \begin{array}{c} 2 \\ 1 \\ 0 \end{array} \\ \begin{array}{c} 1 \\ 1.5 \\ \zeta \end{array} \end{array} \quad \text{and} \quad \beta = \frac{\zeta + \sqrt{\zeta^2 - 1}}{2\sqrt{\zeta^2 - 1}} = \begin{array}{c} \beta \\ \begin{array}{c} 1 \\ 1 \\ 1 \end{array} \\ \begin{array}{c} 1 \\ 1.5 \\ \zeta \end{array} \end{array}$$

As  $\zeta$  grows,

- $\tau_1$  grows with respect to  $\tau_2 \implies y_{\text{tr2}}$  decays faster than  $y_{\text{tr1}}$
  - $\beta$  grows with respect to  $\beta - 1 \implies y_{\text{tr2}}$  becomes smaller than  $y_{\text{tr1}}$
- like in

meaning that  $y_{\text{tr2}}$  may be neglected for large  $\zeta$ 's, say for  $\zeta \geq 1.25$ .

## Step response of overdamped systems (contd)

Rewrite

$$y_{tr}(t) = \overbrace{-k_{st}\beta e^{-t/\tau_1} \mathbb{1}(t)}^{y_{tr1}(t)} + \overbrace{k_{st}(\beta - 1)e^{-t/\tau_2} \mathbb{1}(t)}^{y_{tr2}(t)},$$

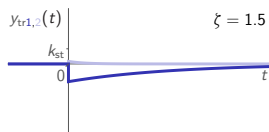
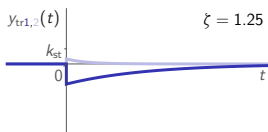
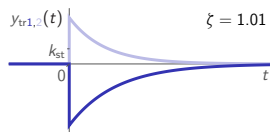
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like in



meaning that  $y_{tr2}$  may be neglected for large  $\zeta$ 's, say for  $\zeta \geq 1.25$ .

## Step response of critically damped systems

If  $\zeta = 1$ , then

$$Y(s) = k_{st} \left( \frac{1}{s} - \frac{1}{s + \omega_n} - \frac{\omega_n}{(s + \omega_n)^2} \right)$$

In the time domain

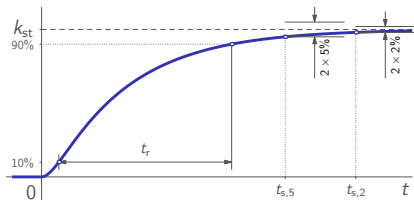
$$y(t) = k_{st} (1 - (1 + \omega_n t) e^{-\omega_n t}) \mathbb{1}(t),$$

with

$$\dot{y}(t) = k_{st} \omega_n^2 t e^{-\omega_n t} \mathbb{1}(t) > 0, \quad \forall t > 0,$$

meaning that  $y$  is monotonically increasing in this case as well.

## Step response of over- and critically damped systems



$$t_r \approx \frac{1}{\omega_n} \begin{cases} 4.9\zeta - 1.5 & \text{if } 1 \leq \zeta < 2 \\ 4.4\zeta - 0.5 & \text{if } \zeta \geq 2 \end{cases}$$

$$t_{s,5} \approx \frac{1}{\omega_n} \begin{cases} -0.6\zeta^2 + 8.5\zeta - 3.1 & \text{if } 1 \leq \zeta < 2 \\ 6\zeta - 0.5 & \text{if } \zeta \geq 2 \end{cases}$$

$$t_{s,2} \approx \frac{1}{\omega_n} \begin{cases} -1.1\zeta^2 + 12.3\zeta - 5.3 & \text{if } 1 \leq \zeta < 2 \\ 7.9\zeta - 0.9 & \text{if } \zeta \geq 2 \end{cases}$$

This response has

- overshoot  $OS = 0\%$  and undershoot  $US = 0\%$
- raise time  $t_r$ 
  - monotonically increases with  $\zeta$
  - is inversely proportional to  $\omega_n$
- settling time  $t_s$ 
  - monotonically increases with  $\zeta$
  - is inversely proportional to  $\omega_n$

monotonic

## Step response of underdamped systems

If  $0 \leq \zeta < 1$ , then

$$\begin{aligned} Y(s) &= k_{\text{st}} \left( \frac{1}{s} - \frac{s + 2\zeta\omega_n}{s^2 + 2\zeta\omega_n s + \omega_n^2} \right) = k_{\text{st}} \left( \frac{1}{s} - \frac{(s + \zeta\omega_n) + \zeta\omega_n}{(s + \zeta\omega_n)^2 + (1 - \zeta^2)\omega_n^2} \right) \\ &= k_{\text{st}} \left( \frac{1}{s} - \frac{1}{\sqrt{1 - \zeta^2}} \frac{\sqrt{1 - \zeta^2}(s + \zeta\omega_n) + \zeta\omega_d}{(s + \zeta\omega_n)^2 + \omega_d^2} \right) \end{aligned}$$

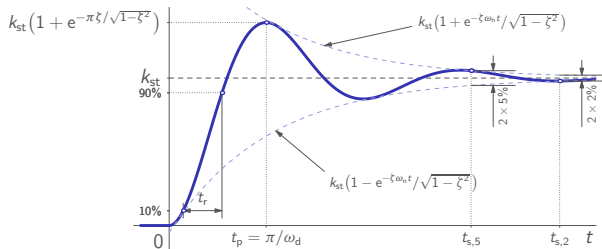
where  $\omega_d := \sqrt{1 - \zeta^2}\omega_n$  is the **damped natural frequency**. Hence,

$$y(t) = k_{\text{st}} \left( 1 - \frac{1}{\sqrt{1 - \zeta^2}} e^{-\zeta\omega_n t} \sin(\omega_d t + \arccos \zeta) \right) \mathbb{1}(t)$$

and it is always within the envelope

$$k_{\text{st}} \left( 1 - \frac{1}{\sqrt{1 - \zeta^2}} e^{-\zeta\omega_n t} \right) \mathbb{1}(t) \leq y(t) \leq k_{\text{st}} \left( 1 + \frac{1}{\sqrt{1 - \zeta^2}} e^{-\zeta\omega_n t} \right) \mathbb{1}(t).$$

## Step response of underdamped systems: transients



$$t_r \approx \frac{1.6\zeta^3 - 0.17\zeta^2 + 0.92\zeta + 1.02}{\omega_n}$$

$$t_{s,5} \approx \frac{2.88}{(-\zeta^2 + 0.64\zeta + 0.96)\zeta\omega_n}$$

$$t_{s,2} \approx \frac{6.49}{(-\zeta^2 + 0.45\zeta + 1.65)\zeta\omega_n}$$

This response has

- overshoot  $OS = e^{-\pi\xi/\sqrt{1-\xi^2}} \cdot 100\%$  and undershoot  $US = 0\%$
- raise time  $t_r$  is inversely proportional to  $\omega_n$
- settling time  $t_s$  is inversely proportional to  $\omega_n$

Qualitatively,

- the smaller  $\zeta$  is, the more oscillatory the response is (larger overshoot, up to  $OS = 100\%$  for  $\zeta = 0$ , and longer  $t_s$ ).

## Step response of underdamped systems: effect of zeros

Let

$$G_\alpha(s) = \frac{k_{st}(\alpha\omega_n s + \omega_n^2)}{s^2 + 2\zeta\omega_n s + \omega_n^2},$$

for  $\alpha \in \mathbb{R}$ . This transfer function is said to have a **zero at  $s = -\omega_n/\alpha$**  since  $G_\alpha(-\omega_n/\alpha) = 0$ . In this case

$$Y_\alpha(s) = \frac{k_{st}(\alpha\omega_n s + \omega_n^2)}{s^2 + 2\zeta\omega_n s + \omega_n^2} \frac{1}{s} = Y_0(s) + \frac{\alpha}{\omega_n} s Y_0(s)$$

and

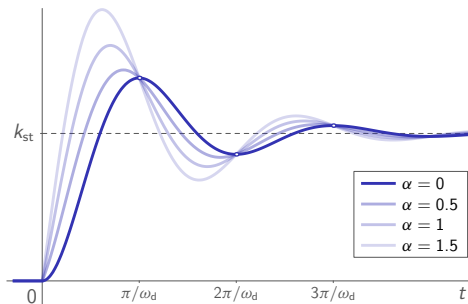
$$y_\alpha(t) = y_0(t) + \frac{\alpha}{\omega_n} \dot{y}_0(t),$$

where  $y_0$  is the response with  $\alpha = 0$  (no zeros) and

$$\frac{\alpha}{\omega_n} \dot{y}_0(t) = \frac{\alpha}{\sqrt{1 - \zeta^2}} \sin(\omega_d t).$$



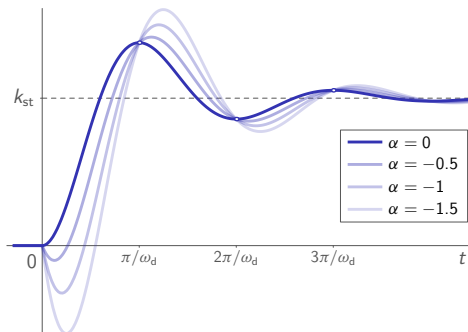
## Step response of underdamped systems: $\alpha > 0$



As  $\alpha$  grows,

- the overshoot OS increases
- the rise time  $t_r$  decreases
- the settling time  $t_s$  increases

## Step response of underdamped systems: $\alpha < 0$



As  $\alpha$  decreases,

- the overshoot OS increases
- the undershoot US increases
- the rise time  $t_r$  decreases
- the settling time  $t_s$  increases