

# Control Theory (035188)

## lecture no. 8

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## Setup

Plant:

$$\begin{cases} \dot{x}(t) = Ax(t) + B(u(t) + d(t)), & x(0) = x_0 \\ y(t) = Cx(t) \\ y_m(t) = Cx(t) + n(t) \end{cases}$$

for known  $A \in \mathbb{R}^{n \times n}$ ,  $B \in \mathbb{R}^n$ , and  $C \in \mathbb{R}^{1 \times n}$  such that  $(A, B)$  is **stabilizable** and  $(C, A)$  is **detectable**.

Uncertainty:

- initial condition  $x_0 \in \mathbb{R}^n$
- load disturbance  $d(t) \in \mathbb{R}$
- measurement noise  $n(t) \in \mathbb{R}$

Control goals:

- stabilize
- reduce the effect of uncertainty on  $x(t)$
- track a known reference signal  $r(t)$  by  $y(t)$

# Outline

State feedback (no uncertainty)

State observer (only the past is uncertain)

Observer-based output feedback (only the past is uncertain)

Effect of disturbances

2DOF state feedback (for curious)

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## State feedback

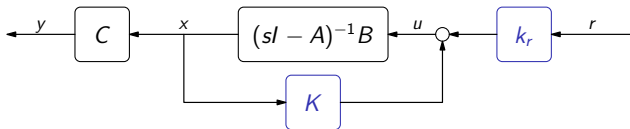
With

$$\begin{cases} \dot{x}(t) = Ax(t) + Bu(t), & x(0) = x_0 \\ y(t) = Cx(t) \end{cases}$$

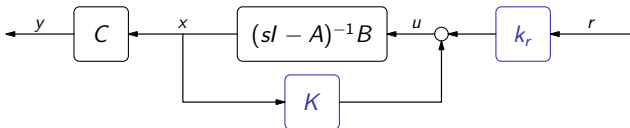
control law

$$u(t) = k_r r(t) + Kx(t)$$

is called **state feedback**. Equivalently,



## State feedback (contd)



The closed-loop state-space realization

$$\begin{cases} \dot{x}(t) = A_K x(t) + B k_r r(t), & x(0) = x_0 \\ y(t) = C x(t) \end{cases}$$

where  $A_K := A + BK$ . The closed-loop transfer function from  $r$  to  $y$ :

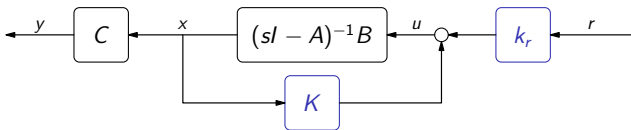
$$T_{yr}(s) = C(sI - A_K)^{-1} B k_r$$

(assuming  $x_0 = 0$ ). Closed-loop characteristic polynomial:

$$\chi_{cl}(s) = \det(sI - A_K).$$

If  $(A, B)$  stabilizable, then  $\chi_{cl}(s)$  can be made Hurwitz by a choice of  $K$ .

## State feedback and system zeros



Let  $P(s) = C(sI - A)^{-1}B = \frac{N_P(s)}{D_P(s)}$ , then

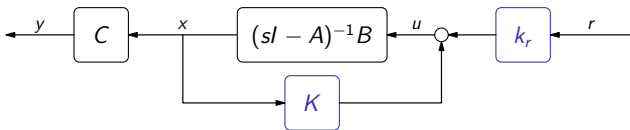
$$u = k_r r + K(sI - A)^{-1}Bu \quad \Longleftrightarrow \quad u = \overbrace{\frac{1}{1 - K(sI - A)^{-1}B}}^{T_{ur}(s)} k_r r.$$

Because  $[1 + C(sI - A)^{-1}B]^{-1} = 1 - C(sI - (A - BC))^{-1}B$ ,

$$\begin{aligned} T_{ur}(s) &= \frac{1}{1 - K(sI - A)^{-1}B} k_r = \frac{D_P(s)}{s^n + \alpha_{n-1}s^{n-1} + \dots + \alpha_1s + \alpha_0} k_r \\ &= (1 + K(sI - A_K)^{-1}B)k_r = \frac{s^n + \beta_{n-1}s^{n-1} + \dots + \beta_1s + \beta_0}{\chi_{cl}(s)} k_r \end{aligned}$$

Hence,  $T_{ur}(s) = k_r D_P(s) / \chi_{cl}(s)$  (mind that  $T_{ur}(\infty) = k_r$ ).

## State feedback and system zeros



Let  $P(s) = C(sI - A)^{-1}B = \frac{N_P(s)}{D_P(s)}$ , then

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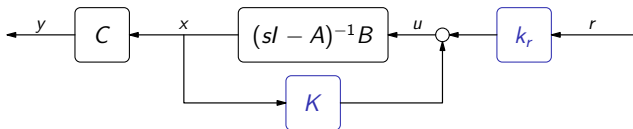
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## State feedback and system zeros (contd)



Thus,

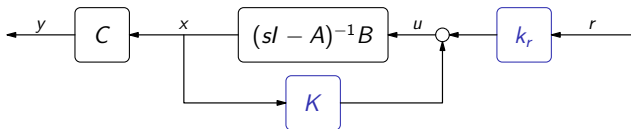
$$T_{yr}(s) = P(s)T_{ur}(s) = \frac{N_P(s) D_P(s)}{D_P(s) \chi_{cl}(s)} k_r = \frac{N_P(s)}{\chi_{cl}(s)} k_r.$$

This means that

→ state feedback does not move zeros

(stable zeros may be canceled by roots of  $\chi_{cl}(s)$  though).

## State feedback and system zeros (contd)



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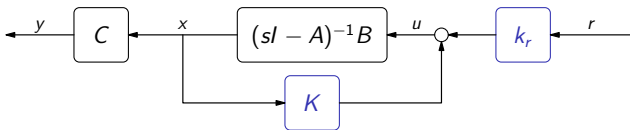
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## State feedback and steady-state error



Steady-state error to a step  $r$ ,

$$e_{ss} := \lim_{t \rightarrow \infty} |r(t) - y(t)| = |1 - T_{yr}(0)| = |1 + CA_K^{-1}Bk_r|.$$

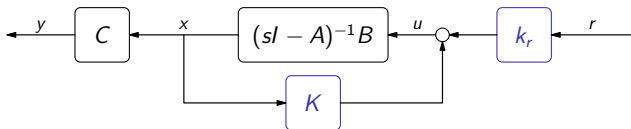
To render it zero, we have to choose

$$k_r = -\frac{1}{CA_K^{-1}B} = -\frac{\chi_d(0)}{N_P(0)}$$

Note that

- $CA_K$  is invertible because it is Hurwitz.
- $k_r$  exists (is finite) iff  $N_P(0) \neq 0$ , i.e. plant has no zeros at the origin

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## Pole placement: companion form

Let's start with  $(A, B)$  in the companion form:

$$A = A_{\text{cf}} := \begin{bmatrix} 0 & 1 & \cdots & 0 \\ \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \cdots & 1 \\ -a_0 & -a_1 & \cdots & -a_{n-1} \end{bmatrix} \quad \text{and} \quad B = B_{\text{cf}} := \begin{bmatrix} 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix}.$$

Select a desired closed-loop characteristic polynomial, say

$$\chi_{\text{cl}}(s) = s^n + \chi_{n-1}s^{n-1} + \cdots + \chi_1s + \chi_0$$

for some coefficients  $\chi_i$ . We already know (Lect. 7) that the state feedback gain

$$K = K_{\text{cf}} := [a_0 - \chi_0 \quad a_1 - \chi_1 \quad \cdots \quad a_{n-1} - \chi_{n-1}]$$

renders  $\chi_{A+BK}(s) = \chi_{\text{cl}}(s)$ .

## Pole placement: arbitrary realization

Conceptually, all we need to do is to

- transform  $(A, B)$  into the companion form by similarity transformation.

Indeed, for any controllable  $(A, B)$  there is (we'll show this by construction) nonsingular  $T$  such that

$$A_{\text{cf}} = TAT^{-1} \quad \text{and} \quad B_{\text{cf}} = TB.$$

Then  $A = T^{-1}A_{\text{cf}}T$ ,  $B = T^{-1}B_{\text{cf}}$ , and

$$K = K_{\text{cf}}T$$

does the job:

$$A + BK = T^{-1}A_{\text{cf}}T + T^{-1}B_{\text{cf}}K_{\text{cf}}T = T^{-1}(A_{\text{cf}} + B_{\text{cf}}K_{\text{cf}})T.$$

Elegant algorithm to construct required  $K_{\text{cf}}T$  w/o explicit calculation of  $T$  is offered by Ackerman's formula.

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Elegant algorithm to construct required  $K_{\text{cf}}T$  w/o explicit calculation of  $T$  is offered by **Ackermann's formula**.

## Ackermann's formula: preliminaries

1. by the Cayley–Hamilton theorem,  $\chi_{\text{ol}}(A_{\text{cf}}) = 0$ , so that

$$\begin{aligned}
 \chi_{\text{cl}}(A_{\text{cf}}) &= A_{\text{cf}}^n + \chi_{n-1}A_{\text{cf}}^{n-1} + \cdots + \chi_1A_{\text{cf}} + \chi_0I \\
 &= -a_{n-1}A_{\text{cf}}^{n-1} - \cdots - a_1A_{\text{cf}} - a_0I \\
 &\quad + \chi_{n-1}A_{\text{cf}}^{n-1} + \cdots + \chi_1A_{\text{cf}} + \chi_0I \\
 &= (\chi_{n-1} - a_{n-1})A_{\text{cf}}^{n-1} + \cdots + (\chi_1 - a_1)A_{\text{cf}} + (\chi_0 - a_0)I.
 \end{aligned}$$

2. if  $e_i$  is the  $i$ th standard basis in  $\mathbb{R}^n$ , then  $\forall i = 1, \dots, n-1$ ,

$$e_i A_{\text{cf}} = e_{i+1} \quad \text{or, equivalently,} \quad e_i^T A_{\text{cf}}^T = e_{i+1}^T.$$

3. if  $A_{\text{cf}} = TAT^{-1}$  and  $B_{\text{cf}} = TB$ , then

$$M_{\text{clcf}} = TM_c \iff T = M_{\text{clcf}}M_c^{-1}.$$



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2. if  $e_i$  is the  $i$ th standard basis in  $\mathbb{R}^n$ , then  $\forall i = 1, \dots, n-1$ ,

$$e'_i A_{cf} = e'_{i+1} \quad \text{or, equivalently,} \quad e'_1 A_{cf}^i = e'_{i+1},$$

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3. if  $A_{\text{cf}} = TAT^{-1}$  and  $B_{\text{cf}} = TB$ , then

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## Ackermann's formula: preliminaries (contd)

### 4. Combining 1 and 2:

$$\begin{aligned}
 K_{cf} &= [a_0 - \chi_0 \quad a_1 - \chi_1 \quad \cdots \quad a_{n-1} - \chi_{n-1}] \\
 &= (a_0 - \chi_0)e'_1 + (a_1 - \chi_1)e'_2 + \cdots + (a_{n-1} - \chi_{n-1})e'_n \\
 &= (a_0 - \chi_0)e'_1 + (a_1 - \chi_1)e'_1 A_{cl} + \cdots + (a_{n-1} - \chi_{n-1})e'_1 A_{cl}^{n-1} \\
 &= -e'_1 \chi_{cl}(A_{cf})
 \end{aligned}$$

### 5. By 2 (and the fact that $B_{cl} = e_n$ ):

$$e'_1 M_{cl} = e'_1 [B_{cl} \quad A_{cl} B_{cl} \quad \cdots \quad A_{cl}^{n-1} B_{cl}] = e'_n$$

## Ackermann's formula: preliminaries (contd)

4. Combining 1 and 2:

$$\begin{aligned}K_{cf} &= [a_0 - \chi_0 \quad a_1 - \chi_1 \quad \cdots \quad a_{n-1} - \chi_{n-1}] \\&= (a_0 - \chi_0)e'_1 + (a_1 - \chi_1)e'_2 + \cdots + (a_{n-1} - \chi_{n-1})e'_n \\&= (a_0 - \chi_0)e'_1 + (a_1 - \chi_1)e'_1 A_{cl} + \cdots + (a_{n-1} - \chi_{n-1})e'_1 A_{cl}^{n-1} \\&= -e'_1 \chi_{cl}(A_{cf})\end{aligned}$$

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$$e'_1 M_{c,cf} = e'_1 [B_{cf} \quad A_{cf} B_{cf} \quad \cdots \quad A_{cf}^{n-1} B_{cf}] = e'_n$$

## Ackermann's formula: derivation

We have:

$$\begin{aligned}K_{cf} &= -e'_1 \chi_{cl}(A_{cf}) && \text{by item 4} \\ &= -e'_1 \chi_{cl}(TAT^{-1}) = -e'_1 T \chi_{cl}(A) T^{-1} \\ &= -e'_1 M_{c,cf} M_c^{-1} \chi_{cl}(A) T^{-1} && \text{by item 3} \\ &= -e'_n M_c^{-1} \chi_{cl}(A) T^{-1} && \text{by item 5}\end{aligned}$$

Now it is time to return to the original coordinates:

$$\begin{aligned}K &= K_{cf} T = -e'_n M_c^{-1} \chi_{cl}(A) T^{-1} T \\ &= -e'_n M_c^{-1} \chi_{cl}(A),\end{aligned}$$

voilà!

## Ackermann's formula

The feedback gain assigning the closed-loop poles to the roots of  $\chi_{cl}(s)$  is

$$K = - [0 \ \cdots \ 0 \ 1] M_c^{-1} \chi_{cl}(A),$$

where

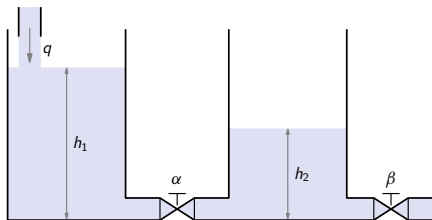
$$M_c = [B \ AB \ \cdots \ A^{n-1}B]$$

and

$$\chi_{cl}(A) = A^n + \chi_{n-1}A^{n-1} + \cdots + \chi_1A + \chi_0I.$$

This gain  $K$  is called **Ackermann's formula** and indeed depends only on the original (controllable) realization.

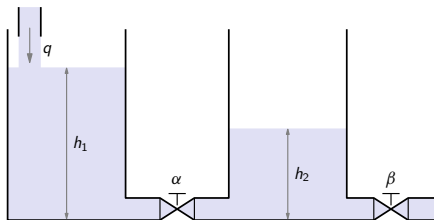
## Example: two-tank system



Here:

- $q$  is the control flow,
- $h_1$  and  $h_2$  are fluid heights,
- $\alpha$  is the resistance to the valve between the tanks,
- $\beta$  is the resistances of the output valve,
- crossing areas of each tank is  $\sigma$ .

## Example: two-tank system (contd)



Its nonlinear dynamics (provided  $h_1 > h_2$ )

$$\sigma \begin{bmatrix} \dot{h}_1(t) \\ \dot{h}_2(t) \end{bmatrix} = \begin{bmatrix} -\alpha \sqrt{h_1(t) - h_2(t)} + q \\ \alpha \sqrt{h_1(t) - h_2(t)} - \beta \sqrt{h_2(t)} \end{bmatrix}$$

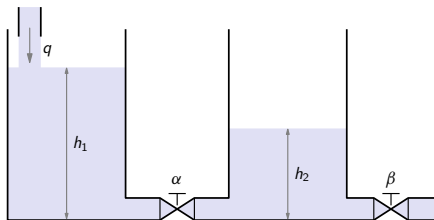
The equilibria are

$$h_{2,\text{eq}} = \frac{\alpha^2}{\alpha^2 + \beta^2} h_{1,\text{eq}} < h_{1,\text{eq}} \quad \text{and} \quad q_{\text{eq}} = \sqrt{\frac{\alpha^2 \beta^2}{\alpha^2 + \beta^2} h_{1,\text{eq}}}$$

for any  $h_{1,\text{eq}} > 0$ .



## Example: two-tank system (contd)



With  $x_i = h_i - h_{i,\text{eq}}$  and  $u = q - q_{\text{eq}}$ , the linearized dynamics are

$$\begin{bmatrix} \dot{x}_1(t) \\ \dot{x}_2(t) \end{bmatrix} = \frac{\alpha \sqrt{1 + (\alpha/\beta)^2}}{2\sigma \sqrt{h_{1,\text{eq}}}} \begin{bmatrix} -1 & 1 \\ 1 & -1 - (\beta/\alpha)^2 \end{bmatrix} \begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix} + \frac{1}{\sigma} \begin{bmatrix} 1 \\ 0 \end{bmatrix} u(t)$$

with negative real poles at

$$s_{1,2} = -\frac{\alpha \sqrt{1 + (\alpha/\beta)^2}}{2\sigma \sqrt{h_{1,\text{eq}}}} \frac{2\alpha^2 + \beta^2 \pm \sqrt{4\alpha^4 + \beta^4}}{2\alpha^2}.$$

## Example: two-tank system (contd)

Let  $\sigma = 1$ ,  $\alpha = \beta = 1$  and choose

$$h_{1,\text{eq}} = 1/2 \quad \implies \quad h_{2,\text{eq}} = 1/4 \quad \text{and} \quad q_{\text{eq}} = 1/2.$$

If  $h_1(0) = h_2(0) = 0$ , then  $x_1(0) = -1/2$  and  $x_2(0) = -1/4$  and we have

$$\begin{bmatrix} \dot{x}_1(t) \\ \dot{x}_2(t) \end{bmatrix} = \begin{bmatrix} -1 & 1 \\ 1 & -2 \end{bmatrix} \begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix} + \begin{bmatrix} 1 \\ 0 \end{bmatrix} u(t), \quad \begin{bmatrix} x_1(0) \\ x_2(0) \end{bmatrix} = - \begin{bmatrix} 1/2 \\ 1/4 \end{bmatrix}.$$

with modes at  $s_1 = -2.618$  and  $s_2 = -0.382$  and

$$P_1(s) = \frac{s + 2}{(s + 2.618)(s + 0.382)} \quad \text{and} \quad P_2(s) = \frac{1}{(s + 2.618)(s + 0.382)}$$

as the transfer functions  $u \mapsto x_1$  and  $u \mapsto x_2$ , respectively.

Our goal is to

regulate  $x$  from  $x(0)$  to  $\lim_{t \rightarrow \infty} x(t) = 0$  in a desired manner (regulator problem), which is effectively the set-point tracking of  $h_{\text{eq}}$ .

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## Example: state feedback

Controllability matrix

$$M_c = \begin{bmatrix} 1 & -1 \\ 0 & 1 \end{bmatrix} \implies M_c^{-1} = \begin{bmatrix} 1 & 1 \\ 0 & 1 \end{bmatrix}.$$

Let's choose

$$\chi_{cl}(s) = s^2 + 2\zeta\omega_n s + \omega_n^2.$$

Ackermann's formula  $K = - [0 \ \dots \ 0 \ 1] M_c^{-1} \chi_{cl}(A)$  reads then

$$\begin{aligned} K &= - [0 \ 1] \begin{bmatrix} 1 & 1 \\ 0 & 1 \end{bmatrix} \left( \begin{bmatrix} -1 & 1 \\ 1 & -2 \end{bmatrix}^2 + 2\zeta\omega_n \begin{bmatrix} -1 & 1 \\ 1 & -2 \end{bmatrix} + \omega_n^2 \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \right) \\ &= - [0 \ 1] \begin{bmatrix} 2 - 2\zeta\omega_n + \omega_n^2 & -3 + 2\zeta\omega_n \\ -3 + 2\zeta\omega_n & 5 - 4\zeta\omega_n + \omega_n^2 \end{bmatrix} \\ &= - [-3 + 2\zeta\omega_n \quad 5 - 4\zeta\omega_n + \omega_n^2] \end{aligned}$$

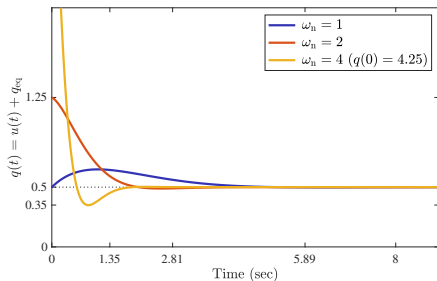
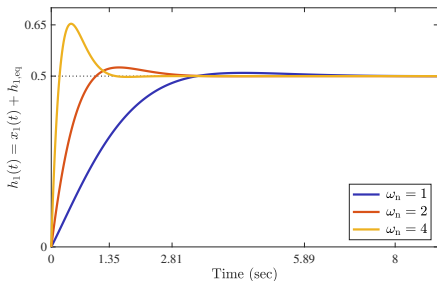
(the absolute values of both components of  $K$  grow with  $\omega_n$ ).

## Example: simulations

The control law

$$u(t) = Kx(t) \implies q(t) = Kx(t) + q_{\text{eq}} - Kh_{\text{eq}}.$$

With  $\sigma = 1$ ,  $\alpha = \beta = 1$ ,  $\zeta = 0.8$ , and  $\omega_n = \{1, 2, 5\}$ ,

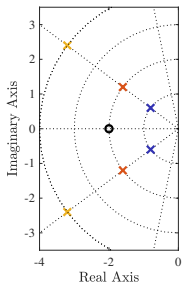
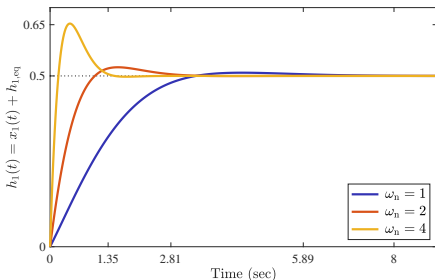


under  $K = - \begin{bmatrix} -1.4 & 2.8 \end{bmatrix}$ ,  $K = - \begin{bmatrix} 0.2 & 2.6 \end{bmatrix}$ ,  $K = - \begin{bmatrix} 3.4 & 8.2 \end{bmatrix}$ . Thus,

- faster poles  $\implies$  faster response & larger control effort
- faster poles  $\implies$  large overshoot (why?)

## Example: simulations (contd)

The reason can be seen in the pole-zero map:



Because system zeros are not moved,

- when poles move left, zero at  $-2$  becomes dominant.

# Outline

State feedback (no uncertainty)

State observer (only the past is uncertain)

Observer-based output feedback (only the past is uncertain)

Effect of disturbances

2DOF state feedback (for curious)

## State reconstruction

Consider state equation

$$\begin{cases} \dot{x}(t) = Ax(t) + Bu(t), & x(0) = x_0, \\ y(t) = Cx(t). \end{cases}$$

If the state vector cannot be measured (this is what typically happens), then it could be reconstructed from measurements of  $y(t)$ . Such reconstructor is called **state observer** or simply **observer**.



## Naïve observer

A possible approach is to construct **virtual plant**, like

$$\dot{\hat{x}}(t) = A\hat{x}(t) + Bu(t), \quad \hat{x}(0) = \hat{x}_0,$$

for some initial guess  $\hat{x}_0$ . Define observation error  $\epsilon(t) := x(t) - \hat{x}(t)$ . Then

$$\dot{\epsilon}(t) = A\epsilon(t), \quad \epsilon(0) = x_0 - \hat{x}_0.$$

Good news:

- if  $A$  is stable,  $\lim_{t \rightarrow \infty} \epsilon(t) = 0$ , i.e.  $\hat{x}(t) \rightarrow x(t)$  asymptotically (no matter what  $u(t)$  is, provided we know it, of course)

Bad news:

- we cannot affect error dynamics,
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## Luenberger observer

Both problems can be resolved by the following modification, which uses  $y$ :

$$\begin{aligned}\dot{\hat{x}}(t) &= A\hat{x}(t) + Bu(t) - L(y(t) - C\hat{x}(t)), & \hat{x}(0) &= \hat{x}_0 \\ &= (A + LC)\hat{x}(t) + Bu(t) - Ly(t), & \hat{x}(0) &= \hat{x}_0\end{aligned}$$

i.e. by adding **correction term** with **observer gain**  $L$ . In this case,

$$\dot{\epsilon}(t) = \underbrace{(A + LC)}_{A_L} \epsilon(t), \quad \epsilon(0) = x_0 - \hat{x}_0.$$

Now we can

- affect its dynamics (more precisely, observable modes of  $(C, A)$ ) and
- stabilize it, provided  $(C, A)$  is detectable.

Although  $y(t) - C\hat{x}(t) = C\epsilon(t)$  depends only on a part of  $\epsilon(t)$ ,

- detectability ensures that  $C\hat{x}(t) \rightarrow y(t) \implies \hat{x}(t) \rightarrow x(t)$ .

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## Transfer functions of Luenberger observer

The observer is a dynamical system having  $u(t)$  and  $y(t)$  as its inputs and  $\hat{x}(t)$  as its output. Under zero initial conditions, the transfer functions from  $u$  to  $\hat{x}$  and from  $y$  to  $\hat{x}$  are (here  $A_L := A + LC$ )

$$G_{\hat{x}u}(s) = (sI - A_L)^{-1}B \quad \text{and} \quad G_{\hat{x}y}(s) = -(sI - A_L)^{-1}L,$$

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respectively.

**Remark:** If we are interested to reconstruct only a part of the state, e.g.

$$z(t) = C_z x(t),$$

the transfer functions from  $u$  to  $\hat{z} := C_z \hat{x}$  and from  $y$  to  $\hat{z}$  are

$$G_{\hat{z}u}(s) = C_z (sI - A_L)^{-1}B \quad \text{and} \quad G_{\hat{z}y}(s) = -C_z (sI - A_L)^{-1}L,$$

respectively.



## Luenberger observer: choice of $L$

Let  $(C, A)$  be observable, then for an arbitrary polynomial

$$\hat{\chi}_{cl}(s) = s^n + \hat{\chi}_{n-1}s^{n-1} + \cdots + \hat{\chi}_1s + \hat{\chi}_0$$

there exists observer gain  $L$  such that  $\hat{\chi}_{cl}(s)$  is characteristic polynomial of observer error, i.e.  $\hat{\chi}_{cl}(s) = \det(sI - A_L)$ .

The choice of  $L$  leading to a required  $\hat{\chi}_d(s)$  is easy if  $(C, A)$  are in the observer canonical form done by the counterpart of Ackermann's formula:

$$L = -\hat{\chi}_d(A)M_0^{-1} \begin{bmatrix} 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix}$$

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---

<sup>1</sup>Consider its derivation a homework assignment.

## Example: two-tank system (contd)

Suppose that fluid height only in the first tank can be measured, i.e.

$$\begin{cases} \begin{bmatrix} \dot{x}_1(t) \\ \dot{x}_2(t) \end{bmatrix} = \begin{bmatrix} -1 & 1 \\ 1 & -2 \end{bmatrix} \begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix} + \begin{bmatrix} 1 \\ 0 \end{bmatrix} u(t), & \begin{bmatrix} x_1(0) \\ x_2(0) \end{bmatrix} = - \begin{bmatrix} 1/2 \\ 1/4 \end{bmatrix} \\ y(t) = \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix} \end{cases}$$

(here  $y = h_1 - h_{1,\text{eq}}$ ). To reconstruct  $x_2(t)$  we build state observer (**virtual sensor**) in the form

$$\begin{bmatrix} \dot{\hat{x}}_1(t) \\ \dot{\hat{x}}_2(t) \end{bmatrix} = \begin{bmatrix} -1 & 1 \\ 1 & -2 \end{bmatrix} \begin{bmatrix} \hat{x}_1(t) \\ \hat{x}_2(t) \end{bmatrix} + \begin{bmatrix} 1 \\ 0 \end{bmatrix} u(t) - \begin{bmatrix} l_1 \\ l_2 \end{bmatrix} (y(t) - \hat{x}_1(t))$$

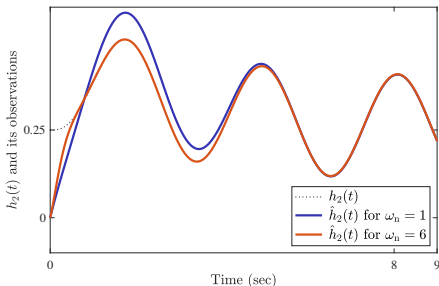
where

$$L = \begin{bmatrix} l_1 \\ l_2 \end{bmatrix} = -\hat{\chi}_{\text{cl}}(A) \begin{bmatrix} 1 & 0 \\ 1 & 1 \end{bmatrix} \begin{bmatrix} 0 \\ 1 \end{bmatrix} = - \begin{bmatrix} -3 + 2\hat{\zeta}\hat{\omega}_n \\ 5 - 4\hat{\zeta}\hat{\omega}_n + \hat{\omega}_n^2 \end{bmatrix}$$

for a desired  $\hat{\chi}_{\text{cl}}(s) = s^2 + 2\hat{\zeta}\hat{\omega}_n s + \hat{\omega}_n^2$ .

## Example: simulations

With  $u(t) = q(t) - q_{\text{eq}} = 0.5 \sin(2t)$ ,  $\hat{\xi} = 0.8$ , and  $\hat{\omega}_n = \{1, 5\}$ ,



under  $L = \begin{bmatrix} 1.4 \\ -2.8 \end{bmatrix}$  and  $L = - \begin{bmatrix} 6.6 \\ 21.8 \end{bmatrix}$ .

# Outline

State feedback (no uncertainty)

State observer (only the past is uncertain)

Observer-based output feedback (only the past is uncertain)

Effect of disturbances

2DOF state feedback (for curious)

## Output feedback: naïve approach

Consider

$$\begin{cases} \dot{x}(t) = Ax(t) + Bu(t), & x(0) = x_0 \\ y(t) = Cx(t) \end{cases}$$

in which the state vector  $x(t)$  is not measured. Therefore, state feedback cannot be used. Under this circumstance, we may try to

- combine state feedback and state observer

instead, i.e. to use observed state in control law as if it were the true state.

This results to the following control law:

$$\begin{cases} \dot{\hat{x}}(t) = A\hat{x}(t) + Bu(t) - L(y(t) - C\hat{x}(t)), & \hat{x}(0) = \hat{x}_0 \\ u(t) = -K\hat{x}(t) + K_r r(t) \end{cases}$$

which called observer-based controller.

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which called **observer-based controller**.

## Observer-based controller

Observer-based control law can be rewritten as

$$\begin{cases} \dot{\hat{x}}(t) = (A + BK + LC)\hat{x}(t) - Ly(t) + Bk_r r(t), & \hat{x}(0) = \hat{x}_0 \\ u(t) = K\hat{x}(t) + k_r r(t) \end{cases}$$

which is a dynamical system having  $y(t)$  and  $r(t)$  as its inputs and  $u(t)$  as its output. Under zero initial conditions, transfer function from  $y$  to  $u$  is

$$C_y(s) = -K(sI - (A + BK + LC))^{-1}L$$

and from  $r$  to  $u$  is

$$C_r(s) = (1 + K(sI - (A + BK + LC))^{-1}B)k_r.$$



## Closed-loop system

State equation of the closed-loop system, from  $r$  to  $y = Cx$  is:

$$\begin{cases} \begin{bmatrix} \dot{x}(t) \\ \dot{\hat{x}}(t) \end{bmatrix} = \begin{bmatrix} A & BK \\ -LC & A + BK + LC \end{bmatrix} \begin{bmatrix} x(t) \\ \hat{x}(t) \end{bmatrix} + \begin{bmatrix} Bk_r \\ Bk_r \end{bmatrix} r(t) \\ y(t) = [C \ 0] \begin{bmatrix} x(t) \\ \hat{x}(t) \end{bmatrix} \end{cases}$$

with initial conditions  $\begin{bmatrix} x_0 \\ \hat{x}_0 \end{bmatrix}$ . Let's now change state vector to

$$\begin{bmatrix} x(t) \\ \epsilon(t) \end{bmatrix} = \begin{bmatrix} I & 0 \\ I & -I \end{bmatrix} \begin{bmatrix} x(t) \\ \hat{x}(t) \end{bmatrix},$$

i.e. apply similarity transformation with

$$T = \begin{bmatrix} I & 0 \\ I & -I \end{bmatrix} = T^{-1}.$$

## Closed-loop system (contd)

We have:

$$\begin{aligned}\tilde{A}_{cl} &= T A_{cl} T^{-1} = \begin{bmatrix} I & 0 \\ I & -I \end{bmatrix} \begin{bmatrix} A & BK \\ -LC & A + BK + LC \end{bmatrix} \begin{bmatrix} I & 0 \\ I & -I \end{bmatrix} \\ &= \begin{bmatrix} A_K & -BK \\ 0 & A_L \end{bmatrix}\end{aligned}$$

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## Closed-loop system (contd)

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$$= \begin{bmatrix} A_K & -BK \\ 0 & A_L \end{bmatrix}$$

$$\tilde{B}_{cl} = T B_{cl} = \begin{bmatrix} I & 0 \\ I & -I \end{bmatrix} \begin{bmatrix} Bk_r \\ Bk_r \end{bmatrix}$$

$$= \begin{bmatrix} Bk_r \\ 0 \end{bmatrix}$$

$$\tilde{C}_{cl} = C_{cl} T^{-1} = [C \ 0] \begin{bmatrix} I & 0 \\ I & -I \end{bmatrix}$$

$$= [C \ 0].$$

## Closed-loop system (contd)

Thus, we end up with the closed-loop system

$$\begin{cases} \begin{bmatrix} \dot{x}(t) \\ \dot{\epsilon}(t) \end{bmatrix} = \begin{bmatrix} A_K & -BK \\ 0 & A_L \end{bmatrix} \begin{bmatrix} x(t) \\ \epsilon(t) \end{bmatrix} + \begin{bmatrix} B \\ 0 \end{bmatrix} k_r r(t), & \begin{bmatrix} x(0) \\ \epsilon(0) \end{bmatrix} = \begin{bmatrix} x_0 \\ x_0 - \hat{x}_0 \end{bmatrix} \\ y(t) = [C \ 0] \begin{bmatrix} x(t) \\ \epsilon(t) \end{bmatrix} \end{cases}$$

Thus, closed-loop characteristic polynomial is

$$\chi_d(s) = \det(sI - A_K) \det(sI - A_L),$$

which is stable provided

- matrix  $A_K = A + BK$  Hurwitz (i.e. state feedback is stabilizing) and
- matrix  $A_L = A + LC$  Hurwitz (i.e. state observer is stable)

(separation principle).

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(**separation principle**).

## Closed-loop transfer function from $r$ to $y$

In

$$\begin{cases} \begin{bmatrix} \dot{x}(t) \\ \dot{\epsilon}(t) \end{bmatrix} = \begin{bmatrix} A_K & -BK \\ 0 & A_L \end{bmatrix} \begin{bmatrix} x(t) \\ \epsilon(t) \end{bmatrix} + \begin{bmatrix} B \\ 0 \end{bmatrix} k_r r(t), & \begin{bmatrix} x(0) \\ \epsilon(0) \end{bmatrix} = \begin{bmatrix} x_0 \\ x_0 - \hat{x}_0 \end{bmatrix} \\ y(t) = [C \ 0] \begin{bmatrix} x(t) \\ \epsilon(t) \end{bmatrix}. \end{cases}$$

all modes of  $A_L$  are **uncontrollable** (check it with PBH). This means that if initial guess  $\hat{x}_0$  is correct, these modes are not excited and can be excluded. Indeed,

$$\dot{\epsilon}(t) = A_L \epsilon(t), \quad \epsilon(0) = x_0 - \hat{x}_0$$

which we already know (observer). Hence, if  $x_0 = \hat{x}_0$ , then  $\epsilon \equiv 0$  and

$$\dot{x}(t) = A_K x(t) - BK \epsilon(t) + B k_r r(t) = A_K x(t) + B k_r r(t), \quad x(0) = x_0$$

is independent on the dynamics of the observer.

## Closed-loop transfer function from $r$ to $y$ (contd)

Another way to see this is via direct calculation of

$$\begin{aligned} T_{yr}(s) &= [C \ 0] \left( s \begin{bmatrix} I & 0 \\ 0 & I \end{bmatrix} - \begin{bmatrix} A_K & -BK \\ 0 & A_L \end{bmatrix} \right)^{-1} \begin{bmatrix} B \\ 0 \end{bmatrix} k_r \\ &= C(sI - A_K)^{-1} B k_r, \end{aligned}$$

which is exactly as in the state-feedback case. It uses the relation

$$\begin{bmatrix} A_{11} & A_{12} \\ 0 & A_{22} \end{bmatrix}^{-1} = \begin{bmatrix} A_{11}^{-1} & -A_{11}^{-1}A_{12}A_{22}^{-1} \\ 0 & A_{22}^{-1} \end{bmatrix}.$$



# Outline

State feedback (no uncertainty)

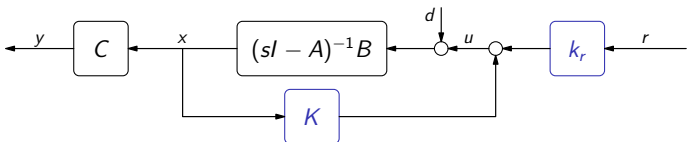
State observer (only the past is uncertain)

Observer-based output feedback (only the past is uncertain)

Effect of disturbances

2DOF state feedback (for curious)

## Disturbance response of state feedback



If  $\dot{x}(t) = Ax(t) + B(u(t) + d(t))$  and  $u(t) = Kx(t)$ , then

$$T_{yd}(s) = C(sI - A_K)^{-1}B$$

The effect of  $K$  is not immediate, although (remember Vieta's formulae)

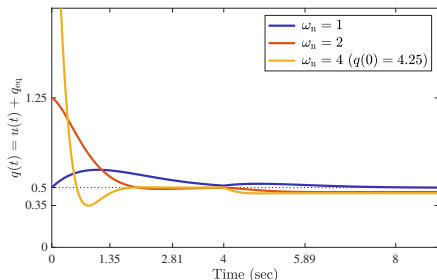
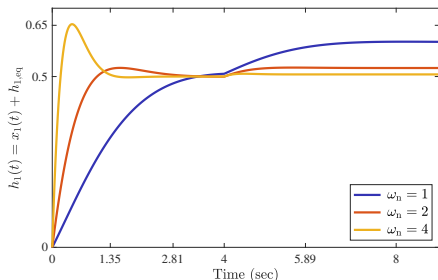
$$T_{yd}(0) = \frac{N_P(0)}{\chi_{cl}(0)} = \frac{N_P(0)}{\prod_i |\lambda_i|}$$

where  $\lambda_i$  are roots of  $\chi_{cl}(s)$ . Hence,

- faster poles  $\implies$  smaller steady-state effects of  $d(t) = \mathbb{1}(t)$

## Example: simulations (contd)

With  $d(t) = 0.05\mathbb{1}(t - 4)$ ,



Thus,

- faster poles  $\implies$  smaller the effect of  $d$

## Luenberger observer and disturbances

If

$$\begin{cases} \dot{x}(t) = Ax(t) + B(u(t) + d(t)), & x(0) = x_0 \\ y(t) = Cx(t) + n(t) \end{cases}$$

the estimator is still

$$\dot{\hat{x}}(t) = A\hat{x}(t) + Bu(t) - L(y(t) - C\hat{x}(t)), \quad \hat{x}(0) = \hat{x}_0$$

(we use all information available), but the estimation error,

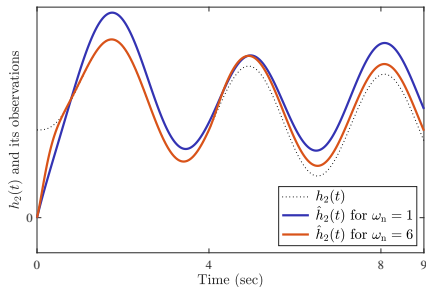
$$\dot{\epsilon}(t) = A_L\epsilon(t) + Bd(t) + Ln(t), \quad \epsilon(0) = x_0 - \hat{x}_0$$

includes both  $d(t)$  and  $n(t)$ .

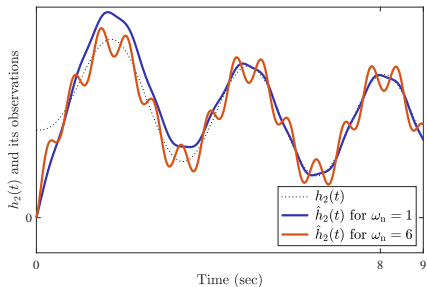
## Example: simulations (contd)

Returning to our two-tank system,

$$d(t) = 0.05\mathbb{1}(t - 4) \text{ and } n(t) = 0$$



$$d(t) = 0 \text{ and } n(t) = 0.025 \sin(10t)$$



and observations no longer converge to  $h_2(t)$ , with

- faster poles  $\implies$  higher gain  $L \implies$  smaller effect of  $d$
- slower poles  $\implies$  lower gain  $L \implies$  smaller effect of  $n$

(but be careful with generalizing that).

## Closed-loop system with observer-based controller

Then the closed-loop system is

$$\begin{cases} \begin{bmatrix} \dot{x}(t) \\ \dot{\epsilon}(t) \end{bmatrix} = \begin{bmatrix} A_K & -BK \\ 0 & A_L \end{bmatrix} \begin{bmatrix} x(t) \\ \epsilon(t) \end{bmatrix} + \begin{bmatrix} B \\ 0 \end{bmatrix} k_r r(t) + \begin{bmatrix} B \\ B \end{bmatrix} d(t) + \begin{bmatrix} 0 \\ L \end{bmatrix} n(t) \\ y(t) = [C \ 0] \begin{bmatrix} x(t) \\ \epsilon(t) \end{bmatrix} \end{cases}$$

with initial conditions  $\begin{bmatrix} x_0 \\ x_0 - \hat{x}_0 \end{bmatrix}$ .

In this case

state-feedback  $T_{yw}(s)$

$$T_{yw}(s) = C(sI - A_K)^{-1} B (1 - K(sI - A_L)^{-1} B)$$

and

$$T_{yn}(s) = -C(sI - A_K)^{-1} BK(sI - A_L)^{-1} L$$

and the effect of  $K$  and  $L$  on the closed-loop behavior is quite complicated.

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with initial conditions  $\begin{bmatrix} x_0 \\ x_0 - \hat{x}_0 \end{bmatrix}$ .

In this case

$$T_{yd}(s) = \overbrace{C(sI - A_K)^{-1}B}^{\text{state-feedback } T_{yd}(s)} (1 - K(sI - A_L)^{-1}B)$$

and

$$T_{yn}(s) = -C(sI - A_K)^{-1}BK(sI - A_L)^{-1}L$$

and the effect of  $K$  and  $L$  on the closed-loop behavior is quite complicated.

# Outline

State feedback (no uncertainty)

State observer (only the past is uncertain)

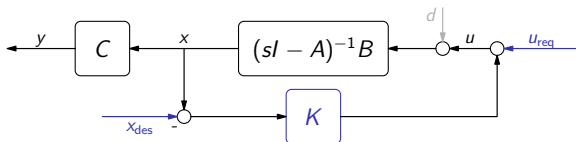
Observer-based output feedback (only the past is uncertain)

Effect of disturbances

2DOF state feedback (for curious)



# Architecture



As usual, two requirements

1.  $x_{des}$  and  $u_{req}$  are bounded
2.  $x_{des} = (sI - A)^{-1}B u_{req}$

stability

consistency

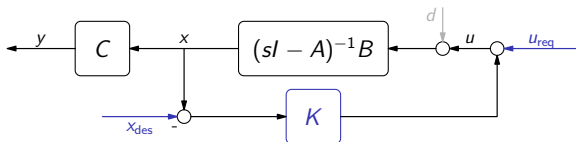
In this case

$$u = u_{req} + K((sI - A)^{-1}Bu - x_{des}) \implies u = \frac{u_{req} - Kx_{des}}{1 - K(sI - A)^{-1}B} = u_{req}$$

and  $x = (sI - A)^{-1}Bu = x_{des}$ , regardless  $K$  (provided it is stabilizing).

$$\text{Generating } x_{des}, y_{des} \rightarrow u_{req} = \frac{y_{des}}{C(sI - A)^{-1}B} \rightarrow x_{des} = (sI - A)^{-1}B u_{req}$$

# Architecture



As usual, two requirements

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2.  $x_{\text{des}} = (sI - A)^{-1} B u_{\text{req}}$

stability  
consistency

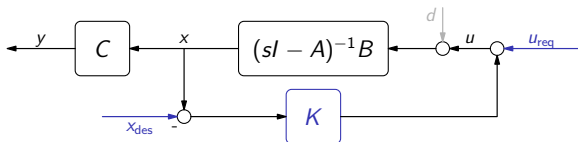
In this case

$$u = u_{\text{req}} + K((sI - A)^{-1} B u - x_{\text{des}}) \implies u = \frac{u_{\text{req}} - K x_{\text{des}}}{1 - K(sI - A)^{-1} B} = u_{\text{req}}$$

and  $x = (sI - A)^{-1} B u = x_{\text{des}}$ , regardless  $K$  (provided it is stabilizing).

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In this case

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Generating  $x_{\text{des}}$ :  $y_{\text{des}} \rightarrow u_{\text{req}} = \frac{y_{\text{des}}}{C(sI - A)^{-1} B} \rightarrow x_{\text{des}} = (sI - A)^{-1} B u_{\text{req}}$

## Example: time-optimal response

We may choose  $y_{\text{des}}$  as the

- fastest filling of tank 1 to  $x_1 = x_{1,f}$  under  $u(t) \in [u_{\text{min}}, u_{\text{max}}]$

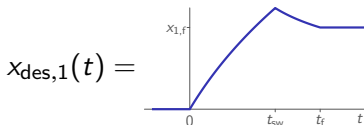
for some  $-q_{\text{eq}} \leq u_{\text{min}} < u_{\text{max}}$ . The optimal bang-bang

$$u_{\text{opt}}(t) = \begin{array}{c} \begin{array}{|c|} \hline u_{\text{max}} \\ \hline \end{array} \\ \begin{array}{|c|} \hline u_{\text{ss}} \\ \hline \end{array} \\ \begin{array}{|c|} \hline u_{\text{min}} \\ \hline \end{array} \end{array} \begin{array}{c} \text{---} \\ \text{---} \\ \text{---} \end{array} \begin{array}{c} t_{\text{sw}} \\ t_f \\ t \end{array}$$

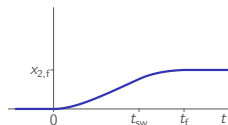
has

$$U_{\text{opt}}(s) = \frac{u_{\text{max}} - (u_{\text{max}} - u_{\text{min}})e^{-st_{\text{sw}}} + (x_{1,f}/P_1(0) - u_{\text{min}})e^{-st_f}}{s},$$

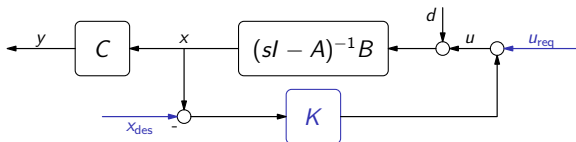
where  $t_{\text{sw}}$  and  $t_f$  are chosen to render  $P_1(s)U_{\text{opt}}(s)$  FIR. In this case



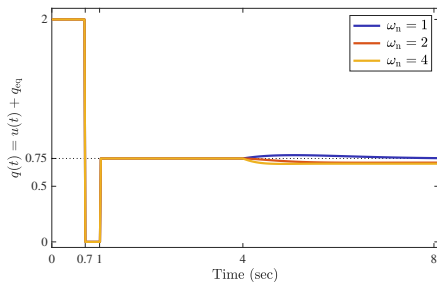
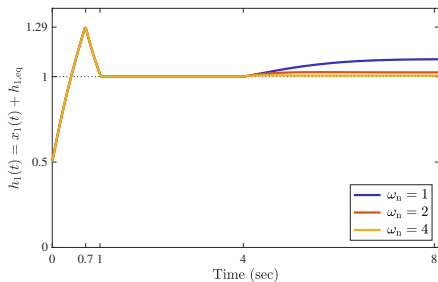
and  $x_{\text{des},2}(t) =$



## Example: simulations (contd)



With  $x_{1,f} = 0.5$ ,  $u_{min} = -0.5 = -q_{eq}$ ,  $u_{max} = 1.5$ , and  $u_{req} = u_{opt}$ ,



where the disturbance  $d(t) = 0.051(t - 4)$ .