Pole placement

Control Theory (00350188) lecture no. 5

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Robust stability

Robust performance

Pole placement



Modeling uncertainty

Robust stability

Robust performance

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Nobody's perfect



In other words, any

 mathematical model is merely a (more / less accurate) approximation of the real world. Robust stability

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- linearization
- unmodeled (high-frequency) dynamics
- parametric drifts
- element failures

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Modeling uncertainty in control systems

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- unmodeled (high-frequency) dynamics
- parametric drifts
- element failures

Modeling uncertainty: DC motor

Consider a DC motor, modeled (from input voltage to shaft velocity) as

$$P(s) = \frac{K_{\rm m} \,\mathrm{e}^{-\tau s}}{(Ls+R)(Js+f) + K_{\rm m}^2}, \tag{1}$$

where K_m is motor constant (= back emf const), R is armature resistance, L is armature inductance, J is load inertia, f is load friction, and the delay τ reflects potential control channel lags (like in digital implementation).



which is an approximation of (1) (which, in turn, is an approximation of the real DC motor).

Moreover, load inertia J might get changed and resistance R is sensitive to thermal conditions (motor heating) and thus also might get changed.

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If L and τ are very small, they are neglected and working model becomes

$$P(s) = \frac{K_{\rm m}}{R(Js+f) + K_{\rm m}^2},\tag{2}$$

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Modeling uncertainty: DC motor (contd)

Possible frequency responses (for some grid over R and J) look then as



where

- $K_{\rm m} = 0.0302$, f = 0.05, $R_0 = 0.316$, and $J_0 = 0.1$ nominal values

$$- 0.9 R_0 \le R \le 1.5 R_0$$
 and $0.8 J_0 \le J \le 1.2 J_0$

uncertain values unmodeled dynamics

and the nominal plant $P_0(s) = \frac{K_{\rm m}}{R_0(J_0s+f)+K_{\rm m}^2}.$

 $-L = 8 \cdot 10^{-5}$ and $\tau = 0.1$

Modeling uncertainty: DC motor (contd)

Thus, at each frequency, frequency response is a region rather than a point:



Frequency-domain modeling

It then does make sense to describe plant frequency response $P(j\omega)$ at each frequency not as a complex number, but rather as set of its possible values

 $P(\mathsf{j}\omega)\in\mathfrak{P}_{\omega}$

where $\mathfrak{P}_{\omega} \subset \mathbb{C}$ is some set for each $\omega \in \mathbb{R}$.

The choice of \mathfrak{P}_{ω} is conceptually nontrivial as

— accurate \mathfrak{P}_{ω} are complicated and hard to deal with in control design,

- easily handleable \mathfrak{P}_{ω} are typically conservative.

In this course (as frequently in engineering), we

sacrifice accuracy for simplicity.

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Multiplicative unstructured uncertainty

Idea: describe \mathfrak{P}_ω as disks in the Nyquist plane around some nominal plant



These disks verify $|P(j\omega) - P_0(j\omega)| \le \ell(\omega)|P_0(j\omega)|$, where

- P_0 is nominal plant (our design model) and

 $-\ell(\omega) \ge 0$ is multiplicative uncertainty radius.

In other words, in this case $\mathfrak{P}_{\omega} = \Big\{ P(j\omega) : \left| \frac{P(j\omega)}{P_0(j\omega)} - 1 \right| \le \ell(\omega) \Big\}.$

DC motor: finding $\ell(\omega)$

To find $\ell(\omega)$, the following steps can be followed: 1. plot $\left|\frac{P(j\omega)}{P_0(j\omega)} - 1\right|$ for different $R \in [0.9R_0, 1.5R_0]$ and $J \in [0.8J_0, 1.2J_0]$;



Typically (like in this case) uncertainty radii $-\ell(\omega)$ are smaller at low frequencies.

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To find $\ell(\omega)$, the following steps can be followed:



Typically (like in this case) uncertainty radii $-\ell(\omega)$ are smaller at low frequencies / larger at high frequencies.

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Typically (like in this case) uncertainty radii $- \ell(\omega)$ are smaller at low frequencies / larger at high frequencies.

DC motor: finding $\ell(\omega)$ (contd)

Now, at each frequency, frequency response is a disk rather than a point:



Disks fully cover actual uncertainty regions, hence

- whatever we can guarantee for disks, holds for the actual motor as well
- but not the other way round (conservatism)

Conservatism may be reduced if a "better" nominal plant $P_0(s)$ is chosen.

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Choice of nominal plant



Might be highly nontrivial, some possible directions:

- place $P_0(j\omega)$ at the center of the minimal covering circle at each ω (might result in very high-order $P_0(s)$, whose handling is too complicated)
- fixed-order "physical" $P_0(s)$ with parameters in the middle of ranges (might not produce the tightest disks, see below)
- fixed-order $P_0(s)$ producing tightest disk (immensely complicated, depends on the choice of tightness measure, etc)

DC motor: choice of nominal plant

Let's pick

$$P_0(s) = P_{0,1}(s) := rac{K_{\mathsf{m}}}{R_1(J_1s + f) + K_{\mathsf{m}}^2}$$

with $R_1 = 1.2R_0 = 0.3792$ and $J_1 = J_0 = 0.1$ chosen as the median of the corresponding intervals $[0.9R_0, 1.5R_0]$ and $[0.8J_0, 1.2J_0]$. This results in



which is not necessarily better than the previous attempt

DC motor: choice of nominal plant (contd)

Consider the following class of nominal plants:

$$P_0(s) = P_{0,2}(s) := rac{K_{\mathsf{m}}}{R_2(J_2s+f) + K_{\mathsf{m}}^2}.$$

Let's aim at placing $P_{0,2}(j\omega)$ to the center of the minimal covering circle of the uncertainty region of $P(j\omega)$ defined by (2) with interval parameters¹ at each ω . Even in this stripped down setting, solution is frequency dependent:



and not always close to the median values $R_1 = 0.3792$ and $J_1 = 0.1$.

¹Note that this P(s) is not the "real" motor in (1)!

DC motor: choice of nominal plant (contd)

For example, if $R_2 = R_2(1) = 0.3424$ and $J_2 = J_2(1) = 0.09694$, we have:



Note that even for $\omega = 1$ we do not have the *minimal* covering circle. This is due to the addition of $L \neq 0$ and $h \neq 0$ to the "real" model.

Multiplicative uncertainty and controller



Let $P(\mathsf{j}\omega)\in\mathfrak{P}_\omega$, where

$$\mathfrak{P}_{oldsymbol{\omega}} = \left\{ P(\mathsf{j}\omega) : \left| rac{P(\mathsf{j}\omega)}{P_0(\mathsf{j}\omega)} - 1
ight| \leq oldsymbol{\ell}(oldsymbol{\omega})
ight\}.$$

Then $L(j\omega) = P(j\omega)C(j\omega) \in \mathfrak{L}_{\omega}$, where

$$\mathfrak{L}_{\omega} = \left\{ L(j\omega) : \left| \frac{L(j\omega)}{L_0(j\omega)} - 1 \right| \leq \ell(\omega) \right\}, \qquad L_0(j\omega) := P_0(j\omega)C(j\omega).$$

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Thus,

- loop multiplicative uncertainty radius does not depend on controller.

Robust stability

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Modeling uncertainty

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Robustness

The ability of a control system to cope with modeling uncertainty (that is, to preserve required characteristics despite uncertainty) is called robustness.

We may talk about

- robust stability
 - (relatively simple problem, we'll discuss it in some technical details)
- robust performance
 - (normally, much harder problem, we'll only see a flavor of this kind of problems)

Robustness

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Robust stability



Let P be such that $P(j\omega) \in \mathfrak{P}_{\omega}$. We say that the

- closed-loop system is robustly stable if it is stable for all $P(j\omega) \in \mathfrak{P}_{\omega}$.

If the system is robustly stable, we say that C robustly stabilizes it.

Robust stability for multiplicative plant uncertainty



Theorem

Let uncertainty be described as

$$\mathfrak{P}_{\omega} = \left\{ P(\mathrm{j}\omega) : \left| rac{P(\mathrm{j}\omega)}{P_0(\mathrm{j}\omega)} - 1
ight| \leq \ell(\omega)
ight\}$$

and all P in this class share the same unstable poles. A controller C then robustly stabilizes the system iff

- 1. C stabilizes nominal plant P_0 and
- 2. $|T_0(j\omega)| < \frac{1}{\ell(\omega)}$, for all ω .²

 $^{^{2}}T_{0}(s) = L_{0}(s)/(1 + L_{0}(s))$ is the nominal complementary sensitivity transfer function.

Robust stability for multiplicative plant uncertainty: proof

When nominal system is stable, we only need to

- ensure that the critical point does not belong to \mathfrak{L}_{ω} for all ω .
- The result then follows by straightforward geometry:


PI controller design for DC motor

Let's now design a PI controller $C(s) = \frac{k_p(s+k_i)}{s}$ for $P_0(s)$. The closed-loop characteristic polynomial $\chi_{cl}(s) = J_0 R_0 s^2 + (K_m^2 + K_m k_p + fR_0)s + K_m k_p k_i$, so the system is stable iff

$$k_{\rm p}k_{\rm i}>0$$
 and $k_{\rm p}>-(K_{\rm m}+fR_0/K_{\rm m})$.

We then choose

- $-k_i$ as the maximal gain for which $|T_0(j\omega)|$ monotonically decreases and
- $-k_{\rm p}$ to achieve a given crossover frequency $\omega_{\rm c}$.

This criterion produces unique coefficients as functions of ω_{c} :

$$k_{\rm p} = \frac{1.0467\sqrt{(\omega_{\rm c}^2 + 0.1398)(\omega_{\rm c}^2 + 0.4195)} - 0.074}{\omega_{\rm c}^2 + 0.2797},$$

$$k_{\rm i} = \frac{\omega_{\rm c}(0.5289\sqrt{(\omega_{\rm c}^2 + 0.1398)(\omega_{\rm c}^2 + 0.4195)} + 0.1398\omega_{\rm c})}{\omega_{\rm c}\sqrt{(\omega_{\rm c}^2 + 0.1398)(\omega_{\rm c}^2 + 0.4195)} - 0.074}$$

(positive iff $\omega_c > 0.24068$, smaller ω_c 's yield undershoot with this strategy).

Comparing $|T_0(j\omega)|$ with $1/\ell(\omega)$ for different ω_c , we get:



Thus, the system is robustly stable only if $\omega_{\rm c} < 11.25$.

The same can be seen through uncertainty disks in the Nyquist plane:



With the borderline $\omega_{\rm c}$ the disk touches the critical point (at $\omega=17.05)$



We see that the actual uncertainty areas (on the right) are also very close to the critical point. This means that the obtained bound on ω_c is virtually non-conservative.

With the borderline $\omega_{\rm c}$ the disk touches the critical point (at $\omega=17.05$)



We see that the actual uncertainty areas (on the right) are also very close to the critical point. This means that the obtained bound on ω_c is virtually non-conservative.

With $P_{0,1}$ as the nominal plan (median nominal R and J), the result is



The largest attainable ω_c is less than 84% of what we obtained with T_0 .

With $P_{0,2}$ as the nominal plan (best fit for $\omega = 1$), the result is



The largest attainable ω_c is less than 96% of what we obtained with T_0 .

choice of "the best" nominal model (design model) is highly nont

With $P_{0,2}$ as the nominal plan (best fit for $\omega = 1$), the result is



The largest attainable ω_c is less than 96% of what we obtained with T_0 . These two examples illustrate the fact that the

choice of "the best" nominal model (design model) is highly nontrivial

Bandwidth limitations due to robust stability

Since $\ell(\omega)$ is typically larger at high frequencies, the condition

$$|T_0(\mathsf{j}\omega)| < \frac{1}{\ell(\omega)}$$

imposes limitations on the achievable closed-loop bandwidth 3 $\omega_{\rm b}.$

 $^{^3\}text{And},$ consequently, on the loop crossover frequency $\omega_{\rm c}.$



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Open-loop control



Let y_0 be the response of the nominal plant P_0 . Then

$$y_0 = P_0 C_{ol} r = T_{ref} r$$
 and hence $y = P C_{ol} r = \frac{P}{P_0} T_{ref} r$

Thus, the normalized control mismatch

$$\frac{|Y(j\omega) - Y_0(j\omega)|}{|R(j\omega)|} = \left|\frac{P(j\omega)}{P_0(j\omega)} - 1\right| |\mathcal{T}_{\mathsf{ref}}(j\omega)| \le \ell(\omega) |\mathcal{T}_{\mathsf{ref}}(j\omega)|$$

and in the frequency range where $T_{
m ref}({
m j}\omega)pprox 1$ (good tracking performance)

control mismatch equals the uncertainty radius of the plant.

In other words,

- open-loop control has no effect on uncertainty.

2DOF control: reference response



With $C_{\rm ol} = T_{\rm ref}/P_0$,

$$y_0 = T_{\text{ref}}r$$
 and $y = \frac{P}{1+PC}\frac{T_{\text{ref}}}{P_0}r + \frac{PC}{1+PC}T_{\text{ref}}r = \frac{T}{T_0}T_{\text{ref}}r$

and the normalized control mismatch

$$\frac{|Y(j\omega) - Y_0(j\omega)|}{|R(j\omega)|} = \left|\frac{T(j\omega)}{T_0(j\omega)} - 1\right| |T_{ref}(j\omega)|$$

depends now upon the uncertainty radius of the complementary sensitivity transfer function (rather than of the plant itself).

2DOF control: disturbance response



In this case,

$$y_0 = T_{d,0}d$$
 and $y = T_dd = \frac{T}{T_0}T_{d,0}d$,

where $T_{d,0} = P_0/(1 + P_0C)$ is the nominal disturbance sensitivity. Then the normalized control mismatch

$$\frac{|Y(j\omega) - Y_0(j\omega)|}{|D(j\omega)|} = \left|\frac{T(j\omega)}{T_0(j\omega)} - 1\right| |T_{d,0}(j\omega)|$$

also depends upon the uncertainty radius of the complementary sensitivity transfer function.

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also depends upon the uncertainty radius of the complementary sensitivity transfer function. What can we say about it ?

Disks mapping under feedback

It can be shown that if the robust stability condition $|T_0(j\omega)| < \frac{1}{\ell(\omega)}$ holds,

$$\left|\frac{\mathcal{T}(\mathsf{j}\omega)}{\mathcal{T}_0(\mathsf{j}\omega)} - 1\right| \leq \ell_{\mathcal{T}_0}(\omega) := \frac{\ell(\omega)}{|1 + L_0(\mathsf{j}\omega)| - \ell(\omega)|L_0(\mathsf{j}\omega)|} = \frac{\ell(\omega)|\mathcal{S}_0(\mathsf{j}\omega)|}{1 - \ell(\omega)|\mathcal{T}_0(\mathsf{j}\omega)|}$$

where $S_0(s) = 1 - T_0(s)$ is the nominal sensitivity function.



Disks mapping under feedback

It can be shown that if the robust stability condition $|T_0(j\omega)| < \frac{1}{\ell(\omega)}$ holds,

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where $S_0(s) = 1 - T_0(s)$ is the nominal sensitivity function.

Remark: As a matter of fact, a disk in the *L*-plane with the center at L_0 is transformed into a *T*-plane disk, whose center is not T_0 , but rather

$$T_{1}(j\omega) = \frac{|1 - \ell^{2}(\omega) T_{0}(j\omega)|^{2}}{1 - \ell^{2}(\omega) |T_{0}(j\omega)|^{2}} \frac{T_{0}(j\omega)}{1 - \ell^{2}(\omega) T_{0}(j\omega)}, \quad \text{with } \ell_{T_{1}}(\omega) = \frac{\ell(\omega) |S_{0}(j\omega)|}{|1 - \ell^{2}(\omega) T_{0}(j\omega)|}$$

(normalized radius). This disk is always *contained* in the disk around T_0 defined above and its non-normalized radius is

$$\ell_{T_1}(\omega)|T_1(j\omega)| = \frac{\ell(\omega)|S_0(j\omega)|}{1-\ell(\omega)|T_0(j\omega)|} \frac{|T_0(j\omega)|}{1+\ell(\omega)|T_0(j\omega)|} \leq \ell_{T_0}(\omega)|T_0(j\omega)|.$$

But the use of T_1 as the nominal T for controller design might not be easy (complexity).

Disks mapping by feedback: what does it mean?



The relation $\ell_{\mathcal{T}_0}(\omega) = rac{\ell(\omega)}{|1+L_0(j\omega)|-\ell(\omega)|L_0(j\omega)|}$ effectively says then that

- feedback reduces uncertainty level at frequencies ω , where the disk \mathfrak{L}_{ω} is at a distance of at least 1 from the critical point and that
- the further \mathfrak{L}_{ω} from the critical point -1 + j0, the lower the uncertainty level in $T(j\omega)$ is (provided we pick T_0 as the nominal T, of course)

— uncertainty is always aggravated by feedback at ω 's where $|S_0(j\omega)|>1$

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- the further \mathfrak{L}_{ω} from the critical point -1 + j0, the lower the uncertainty level in $T(j\omega)$ is (provided we pick T_0 as the nominal T, of course)

Also, the relation $\ell_{T_0}(\omega) = \frac{\ell(\omega)|S_0(j\omega)|}{1-\ell(\omega)|T_0(j\omega)|}$ implies that

– uncertainty is always aggravated by feedback at ω 's where $|S_0({
m j}\omega)|>1$

Robust performance of PI controlled DC motor



In all 3 cases $\ell_{T_0}(0) = 0$, which is the result of the use of an integral action in the controller (as then $S_0(0) = 0$, while $\ell(0)|T_0(0)| = 0.2676 < 1$).

Robust performance of PI controlled DC motor



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Robust performance: DC motor comparison

Let us choose $T_{\text{ref}}(s) = \frac{\omega_N^2}{s^2 + \sqrt{2}\omega_N s + \omega_N^2}$ with $\omega_N = \frac{2}{3}$ and compare 2 strategies discussed in the beginning of this section. The feedback controller is the PI discussed above with $\omega_c = 4$. Advantages of feedback are clear:



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Modal analysis: idea



Express closed-loop performance requirements

- in terms of the location of closed-loop poles

which are roots of the characteristic polynomial

$$\chi_{\mathsf{cl}}(s) := N_P(s)N_C(s) + D_P(s)D_C(s),$$

where

$$P(s) = \frac{N_P(s)}{D_P(s)}$$
 and $C(s) = \frac{N_C(s)}{D_C(s)}$

and deg $\chi_{cl}(s) = \deg D_P(s) + \deg D_C(s)$ (assuming that P(s) and C(s) are proper and there are no pole / zero cancellations between P(s) and C(s)).

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Modal analysis: idea (contd)

Example:



small overshoot

- precise for 2-order systems w/o zeros
- justified for systems with 2-order dominant dynamics

Pole placement

Modal analysis: idea (contd)

Example:



- precise for 2-order systems w/o zeros
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Modal analysis: idea (contd)

Example:



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Example: static controller

Let $P(s) = 1/(s^2 + 2s)$ and controller is of the form $C(s) = \beta_0$. Then

$$\chi_{\mathsf{cl}}(s) = s^2 + 2s + \beta_0.$$

Closed-loop poles can only be placed to points on root-locus branches:

Pole placement

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Closed-loop poles can only be placed to points on root-locus branches:



Example: 1-order strictly proper controller

Let $P(s) = 1/(s^2 + 2s)$ and controller is of the form $C(s) = \frac{\beta_0}{\alpha_1 s + \alpha_0}$. Then

$$\chi_{cl}(s) = \alpha_1 s^3 + (\alpha_0 + 2\alpha_1)s^2 + 2\alpha_0 s + \beta_0$$

= $\chi_3 s^3 + \chi_2 s^2 + \chi_1 s + \chi_0.$

Still constrained: $\chi_1 - 2\chi_2 + 4\chi_3 = 0$.



which cannot be solved for arbitrarily χ_i .

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= $\chi_3 s^3 + \chi_2 s^2 + \chi_1 s + \chi_0.$

Still constrained: $\chi_1-2\chi_2+4\chi_3=0.$ Alternative form:

$$egin{bmatrix} 1 & 0 & 0 \ 2 & 1 & 0 \ 0 & 2 & 0 \ 0 & 0 & 1 \end{bmatrix} egin{bmatrix} lpha_1 \ lpha_0 \ eta_0 \end{bmatrix} = egin{bmatrix} \chi_3 \ \chi_2 \ \chi_1 \ \chi_0 \end{bmatrix},$$

which cannot be solved for arbitrarily χ_i .

Example: 1-order bi-proper controller

Let $P(s) = 1/(s^2 + 2s)$ and controller is of the form $C(s) = \frac{\beta_1 s + \beta_0}{\alpha_1 s + \alpha_0}$. Then

$$egin{aligned} \chi_{\mathsf{cl}}(s) &= lpha_1 s^3 + (lpha_0 + 2 lpha_1) s^2 + (eta_1 + 2 lpha_0) s + eta_0 \ &= \chi_3 s^3 + \chi_2 s^2 + \chi_1 s + \chi_0. \end{aligned}$$

Unconstrained, χ_i can be arbitrary.

$$\begin{bmatrix} 1 & 0 & 0 & 0 \\ 2 & 1 & 0 & 0 \\ 0 & 2 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} \alpha_1 \\ \alpha_0 \\ \beta_1 \\ \beta_0 \end{bmatrix} = \begin{bmatrix} \chi_3 \\ \chi_2 \\ \chi_1 \\ \chi_0 \end{bmatrix}$$

which can be solved for arbitrarily χ_i as det $M_{\rm S} = 1 \neq 0$.

Example: 1-order bi-proper controller

Let $P(s) = 1/(s^2 + 2s)$ and controller is of the form $C(s) = \frac{\beta_1 s + \beta_0}{\alpha_1 s + \alpha_0}$. Then

$$\chi_{cl}(s) = lpha_1 s^3 + (lpha_0 + 2lpha_1) s^2 + (eta_1 + 2lpha_0) s + eta_0$$

= $\chi_3 s^3 + \chi_2 s^2 + \chi_1 s + \chi_0.$

Unconstrained, χ_i can be arbitrary. Alternative form:

$$\underbrace{\begin{bmatrix} 1 & 0 & 0 & 0 \\ 2 & 1 & 0 & 0 \\ 0 & 2 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}}_{M_{\mathrm{S}}} \begin{bmatrix} \alpha_{1} \\ \alpha_{0} \\ \beta_{1} \\ \beta_{0} \end{bmatrix} = \begin{bmatrix} \chi_{3} \\ \chi_{2} \\ \chi_{1} \\ \chi_{0} \end{bmatrix},$$

which can be solved for arbitrarily χ_i as det $M_S = 1 \neq 0$.

Example: 2-order strictly proper controller

Let $P(s) = 1/(s^2 + 2s)$ and controller is of the form $C(s) = \frac{\beta_1 s + \beta_0}{\alpha_2 s^2 + \alpha_1 s + \alpha_0}$. Then

$$egin{aligned} \chi_{\mathsf{cl}}(s) &= lpha_2 s^4 + (lpha_1 + 2 lpha_2) s^3 + (lpha_0 + 2 lpha_1) s^2 + (eta_1 + 2 lpha_0) s + eta_0 \ &= \chi_4 s^4 + \chi_3 s^3 + \chi_2 s^2 + \chi_1 s + \chi_0. \end{aligned}$$

Unconstrained, χ_i can be arbitrary, which is seen from

$$\underbrace{\begin{bmatrix} 1 & 0 & 0 & 0 & 0 \\ 2 & 1 & 0 & 0 & 0 \\ 0 & 2 & 1 & 0 & 0 \\ 0 & 0 & 2 & 1 & 0 \\ 0 & 0 & 0 & 0 & 1 \end{bmatrix}}_{M_{\rm S}} \begin{bmatrix} \alpha_2 \\ \alpha_1 \\ \alpha_0 \\ \beta_1 \\ \beta_0 \end{bmatrix} = \begin{bmatrix} \chi_4 \\ \chi_3 \\ \chi_2 \\ \chi_1 \\ \chi_0 \end{bmatrix}$$

(always solvable in χ_i as det $M_S = 1 \neq 0$).

Example: what can we learn from it

- controlers of sufficient high order needed for arbitrary pole placement
- polynomial equations reduce to linear equations

Preliminary: multiplication of polynomials

Let $A(s) = a_n s^n + \cdots + a_1 s + a_0$ and $B(s) = b_m s^m + \cdots + b_1 s + b_0$ with $n \ge m$, so that

$$C(s) := A(s)B(s) = c_{n+m}s^{n+m} + c_{n+m-1}s^{n+m-1} + \cdots + c_1s + c_0.$$

The coefficients of C(s) can be calculated from the table

	a _n s ⁿ	$a_{n-1}s^{n-1}$	• • •	a_1s	a_0
b _m s ^m	a _n b _m s ^{n+m}	$a_{n-1}b_ms^{n+m-1}$	• • •	$a_1 b_m s^{m+1}$	$a_0 b_m s^m$
$b_{m-1}s^{m-1}$	$a_n b_{m-1} s^{n+m-1}$	$a_{n-1}b_{m-1}s^{n+m-2}$	• • •	$a_1b_{m-1}s^m$	$a_0 b_{m-1} s^{m-1}$
$b_{m-2}s^{m-2}$	$a_n b_{m-2} s^{n+m-2}$	$a_{n-1}b_{m-2}s^{n+m-3}$	• • •	$a_1b_{m-2}s^{m-1}$	$a_0b_{m-2}s^{m-2}$
:	:	:		:	:
$b_{r}c^{2}$	$\frac{1}{2} b_{n} c^{n+2}$	$b = c^{n+1}$		$\frac{1}{2}$	$\frac{1}{2}$
025	$a_n D_2 S$	$a_{n-1}b_{2}s$		a1025	a0025
b_1s	$a_nb_1s^{n+1}$	$a_{n-1}b_1s^n$	• • •	$a_1b_1s^2$	$a_0 b_1 s$
b_0	a _n b ₀ s ⁿ	$a_{n-1}b_0s^{n-1}$	• • •	a_1b_0s	$a_0 b_0$

by summing up elements on each anti-diagonal.

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	a _n s ⁿ	$a_{n-1}s^{n-1}$	•••	a_1s	<i>a</i> 0
b _m s ^m	a _n b _m s ^{n+m}	$a_{n-1}b_ms^{n+m-1}$	• • •	$a_1 b_m s^{m+1}$	$a_0 b_m s^m$
$b_{m-1}s^{m-1}$	$a_n b_{m-1} s^{n+m-1}$	$a_{n-1}b_{m-1}s^{n+m-2}$		$a_1b_{m-1}s^m$	$a_0 b_{m-1} s^{m-1}$
$b_{m-2}s^{m-2}$	$a_n b_{m-2} s^{n+m-2}$	$a_{n-1}b_{m-2}s^{n+m-3}$		$a_1b_{m-2}s^{m-1}$	$a_0b_{m-2}s^{m-2}$
:	:	:		:	:
b. c ²	$a \ b \ c^{n+2}$	$b c^{n+1}$		$a b c^3$	$\frac{1}{2}$
D_2S	a _n D ₂ S	$a_{n-1}b_2s$		a1025	$a_0 D_2 S$
b_1s	$a_nb_1s^{n+1}$	$a_{n-1}b_1s^n$	• • •	$a_1b_1s^2$	$a_0 b_1 s$
b_0	a _n b ₀ s ⁿ	$a_{n-1}b_0s^{n-1}$	• • •	$a_1 b_0 s$	$a_0 b_0$

by summing up elements on each anti-diagonal.
Preliminary: multiplication of polynomials (contd)

This results in the following formula for coefficients of C(s):



Pole placement

Sylvester matrix

Let (here $a_n \neq 0$)

 $D_P(s) = a_n s^n + \cdots + a_1 s + a_0$ and $N_P(s) = b_n s^n + \cdots + b_1 s + b_0$.

The $(2n+1) \times (2n+2)$ matrix

$$M_{\mathsf{S}} := \begin{bmatrix} M_{a,n} & M_{b,n} \end{bmatrix} = \begin{bmatrix} a_n & 0 & \cdots & 0 & b_n & 0 & \cdots & 0 \\ a_{n-1} & a_n & \cdots & 0 & b_{n-1} & b_n & \cdots & 0 \\ \vdots & \vdots & \ddots & \vdots & \vdots & \vdots & \ddots & \vdots \\ a_0 & a_1 & \cdots & a_n & b_0 & b_1 & \cdots & b_n \\ 0 & a_0 & \cdots & a_{n-1} & 0 & b_0 & \cdots & b_{n-1} \\ \vdots & \vdots & \ddots & \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \cdots & a_0 & 0 & 0 & \cdots & b_0 \end{bmatrix}$$

called Sylvester matrix, associated with $D_P(s)$ and $N_P(s)$.

Sylvester matrix (contd)

We need also some sub-matrices of M_{S} :

- $M_{\rm S1}$ is the $(2n+1) \times (2n+1)$ matrix obtained from $M_{\rm S}$ by eliminating its (n+2)th column
- $M_{\rm S2}$ is the $2n \times 2n$ matrix obtained from $M_{\rm S}$ by eliminating its 1st row and 1st and (n+2)th columns

That is:

$$M_{S1} := \begin{bmatrix} a_n & 0 & \cdots & 0 & 0 & \cdots & 0 \\ a_{n-1} & a_n & \cdots & 0 & b_n & \cdots & 0 \\ \vdots & \vdots & \ddots & \vdots & \vdots & \ddots & \vdots \\ a_0 & a_1 & \cdots & a_n & b_1 & \cdots & b_n \\ 0 & a_0 & \cdots & a_{n-1} & b_0 & \cdots & b_{n-1} \\ \vdots & \vdots & \ddots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \cdots & a_0 & 0 & \cdots & b_0 \end{bmatrix}$$

and M_{S2} is in green.

Sylvester's theorem

Theorem

Polynomials $D_P(s)$ and $N_P(s)$ relatively prime iff the associated Sylvester matrix M_S has full (row) rank.

Corollary

 $D_P(s)$ and $N_P(s)$ relatively prime iff det $M_{S1} \neq 0$ (or det $M_{S2} \neq 0$).

Example: Let $D_P(s) = s(s+2)$ and $N_P(s) = s+2$. Then

$$M_{\rm S1} = \begin{bmatrix} 1 & 0 & 0 & 0 & 0 \\ 2 & 1 & 0 & 0 & 0 \\ 0 & 2 & 1 & 1 & 0 \\ 0 & 0 & 2 & 2 & 1 \\ 0 & 0 & 0 & 0 & 2 \end{bmatrix}$$

is indeed singular (and so is $M_{
m S2}$) as its 3rd and 4th columns coincide.

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is indeed singular (and so is M_{S2}) as its 3rd and 4th columns coincide.

Pole placement: *n*-order controller

Let P(s) have (irreducible) order n and consider n-order controller:

$$C(s) = \frac{\beta_n s^n + \dots + \beta_1 s + \beta_0}{\alpha_n s^n + \dots + \alpha_1 s + \alpha_0}$$

This yields 2*n*-order $\chi_{cl}(s) = \chi_{2n}s^{2n} + \cdots \chi_1s + \chi_0$ satisfying



-2n+1 equations, 2n+2 variables, full-rank $M_S \implies \infty$ many solutions

Pole placement: (n-1)-order controller

Let's try to reduce the order of the controller to n - 1. This implies:

$$\alpha_n = \beta_n = \chi_{2n} = 0$$

and then:



− 2*n* equations, 2*n* variables, det $M_S \neq 0 \implies$ unique solution

Pole placement: (n-1)-order controller

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and then:



- 2*n* equations, 2*n* variables, det $M_{\rm S} \neq 0 \implies$ unique solution

any further reduction impossible (more equations than variables)

n-order controller: exploiting freedom we have

We have one "spare" variable in this case, which can be exploited to

bring about additional properties to the controller.

For example, we may enforce $\beta_n = 0$ (strictly proper controller). Then:

$$\underbrace{\begin{bmatrix} a_{n} & 0 & \cdots & 0 & 0 & \cdots & 0 \\ a_{n-1} & a_{n} & \cdots & 0 & b_{n} & \cdots & 0 \\ \vdots & \vdots & \ddots & \vdots & \vdots & \ddots & \vdots \\ a_{0} & a_{1} & \cdots & a_{n} & b_{1} & \cdots & b_{n} \\ 0 & a_{0} & \cdots & a_{n-1} & b_{0} & \cdots & b_{n-1} \\ \vdots & \vdots & \ddots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \cdots & a_{0} & 0 & \cdots & b_{0} \end{bmatrix}}_{M_{S1}} \begin{bmatrix} \alpha_{n} \\ \alpha_{n-1} \\ \vdots \\ \alpha_{0} \\ \beta_{n-1} \\ \vdots \\ \beta_{0} \end{bmatrix} = \begin{bmatrix} \chi_{2n} \\ \chi_{2n-1} \\ \vdots \\ \chi_{n+1} \\ \chi_{n} \\ \vdots \\ \chi_{0} \end{bmatrix}$$

-2n+1 equations, 2n+1 variables, det $M_{S1} \neq 0 \implies$ unique solution

n-order controller: exploiting freedom we have (contd)

Another possibility is to enforce $\alpha_0 = 0$ (integral action). Then:



-2n+1 equations, 2n+1 variables, det $M_{S3} \neq 0 \implies$ unique solution (the non-singularity of M_{S3} can be proved under condition that $b_0 \neq 0$).

Pole-placement as a design tool

Pros:

- ∴ arbitrary pole placement
- $\ddot{\}$ easily computable

Cons:

- $\stackrel{_\sim}{_\sim}$ (almost) no control over controller poles
- 🐥 no control over controller zeros
- 🐥 no dominance guarantees

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